# Solution Manual for Statics and Mechanics of Materials 2nd Edition Beer Johnston DeWolf Mazurek 00733981609780073398167 

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## SOLUTION

(a) Parallelogram law:

(b) Triangle rule:


We measure:
$R=1391 \mathrm{kN}, \alpha=47.8^{\circ}$
$\mathbf{R}=1391 \mathrm{~N} \boldsymbol{\sim}^{\boldsymbol{*}} 47.8^{\circ}$

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## SOLUTION

(a) Parallelogram law:

(b) Triangle rule:


We measure:

$$
R=906 \mathrm{lb}, \quad \alpha=26.6^{\circ}
$$

$R=906 \mathrm{lb}$ < $26.6^{\circ}$

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## SOLUTION

(a) Parallelogram law:

(b) Triangle rule:


We measure:

$$
R=20.1 \mathrm{kN}, \quad \alpha=21.2^{\circ}
$$

$$
\mathbf{R}=20.1 \mathrm{kN} \square 21.2^{\circ}
$$

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## SOLUTION

(a) Parallelogram law:

(b) Triangle rule:


We measure:
$R=8.03 \mathrm{kips}, \alpha=3.8^{\circ}$
$\mathbf{R}=8.03 \mathrm{kips} \not \supset 3.8^{\circ} \boldsymbol{\triangleleft}$

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## SOLUTION

(a) Using the triangle rule and law of sines:

$$
\begin{aligned}
\frac{\sin \beta}{240 \mathrm{lb}} & =\frac{\sin 60^{\circ}}{300 \mathrm{lb}} \\
\sin \beta & =0.69282 \\
\beta & =43.854^{\circ} \\
\alpha+\beta+60^{\circ} & =180^{\circ} \\
\alpha & =180^{\circ}-60^{\circ}-43.854^{\circ} \\
& =76.146^{\circ}
\end{aligned}
$$


(b) Law of sines:

$$
\frac{F_{b b^{\prime}}}{\sin 76.146^{\circ}}=\frac{300 \mathrm{lb}}{\sin 60^{\circ}}
$$

$$
F_{b b^{\prime}}=336 \mathrm{lb}
$$



## PROBLEM 2.6

The $300-\mathrm{lb}$ force is to be resolved into components along lines $a-a^{\prime}$ and $b-b^{\prime}$. (a) Determine the angle $\alpha$ by trigonometry knowing that the component along line $b-b^{\prime}$ is to be 120 lb . (b) What is the corresponding value of the component along $a-a^{\prime}$ ?

## SOLUTION

Using the triangle rule and law of sines:
(a)

$$
\frac{\sin \alpha}{120 \mathrm{lb}}=\frac{\sin 60^{\circ}}{300 \mathrm{lb}}
$$



$$
\sin \alpha=0.34641
$$

$$
\alpha=20.268^{\circ}
$$

(b)

$$
\begin{aligned}
\alpha+\beta+60^{\circ} & =180^{\circ} \\
\beta & =180^{\circ}-60^{\circ}-20.268^{\circ} \\
& =99.732^{\circ} \\
\frac{F_{a a^{\prime}}}{\sin 99.732^{\circ}} & =\frac{300 \mathrm{lb}}{\sin 60^{\circ}}
\end{aligned} \quad F_{a a^{\prime}}=341 \mathrm{lb}
$$



## PROBLEM 2.7

A trolley that moves along a horizontal beam is acted upon by two forces as shown. (a) Knowing that $\alpha=25^{\circ}$, determine by trigonometry the magnitude of the force $\mathbf{P}$ so that the resultant force exerted on the trolley is vertical. (b) What is the corresponding magnitude of the resultant?

## SOLUTION



Using the triangle rule and the law of sines:
(a)

$$
\frac{1600 \mathrm{~N}}{\sin 25^{\circ}}=\frac{P}{\sin 75^{\circ}}
$$

$$
P=3660 \mathrm{~N} 4
$$

(b)

$$
\begin{aligned}
25^{\circ}+\beta+75^{\circ} & =180^{\circ} \\
\beta & =180^{\circ}-25^{\circ}-75^{\circ} \\
& =80^{\circ} \\
\frac{1600 \mathrm{~N}}{\sin 25^{\circ}} & =\frac{R}{\sin 80^{\circ}} \quad R=3730 \mathrm{~N}
\end{aligned}
$$



## SOLUTION



Using the law of sines:

$$
\begin{aligned}
\frac{T_{A C}}{\sin 30^{\circ}} & =\frac{R}{\sin 125^{\circ}}=\frac{2.2 \mathrm{kN}}{\sin 25} \\
T_{A C} & =2.603 \mathrm{kN} \\
R & =4.264 \mathrm{kN}
\end{aligned}
$$

(a)
(b)

$$
\begin{aligned}
T_{A C} & =2.60 \mathrm{kN} \\
R & =4.26 \mathrm{kN}
\end{aligned}
$$



## SOLUTION

Using the triangle rule and law of sines:
(a)

$$
\begin{aligned}
\frac{\sin \alpha}{50 \mathrm{~N}} & =\frac{\sin 25^{\circ}}{35 \mathrm{~N}} \\
\sin \alpha & =0.60374 \\
\alpha & =37.138^{\circ}
\end{aligned}
$$


(b)

$$
\begin{aligned}
\alpha+\beta+25^{\circ} & =180^{\circ} \\
\beta & =180^{\circ}-25^{\circ}-37.138^{\circ} \\
& =117.862^{\circ} \\
\frac{R}{\sin 117.862^{\circ}} & =\frac{35 \mathrm{~N}}{\sin 25^{\circ}} \quad R=73.2 \mathrm{~N}
\end{aligned}
$$



## SOLUTION



Using the law of cosines:

$$
\begin{aligned}
T_{A C}{ }^{2} & =(3 \mathrm{kN})^{2}+(4.8 \mathrm{kN})^{2}-2(3 \mathrm{kN})(4.8 \mathrm{kN}) \cos 30^{\circ} \\
T_{A C} & =2.6643 \mathrm{kN}
\end{aligned}
$$

Using the law of sines:

$$
\begin{aligned}
\frac{\sin \alpha}{3 \mathrm{kN}} & =\frac{\sin 30^{\circ}}{2.6643 \mathrm{kN}} \\
\alpha & =34.3^{\circ}
\end{aligned}
$$

$$
\mathbf{T}_{A C}=2.66 \mathrm{kN}\left\ulcorner 34.3^{\circ}\right.
$$



## SOLUTION



Using the law of cosines:

$$
\begin{aligned}
P^{2} & =(1600 \mathrm{~N})^{2}+(2500 \mathrm{~N})^{2}-2(1600 \mathrm{~N})(2500 \mathrm{~N}) \cos 75^{\circ} \\
P & =2596 \mathrm{~N}
\end{aligned}
$$

Using the law of sines:

$$
\underline{\sin \alpha}=\sin 75^{\circ}
$$

$$
1600 \mathrm{~N} \quad 2596 \mathrm{~N}
$$

$$
\alpha=36.5^{\circ}
$$

$P$ is directed $90^{\circ}-36.5^{\circ}$ or $53.5^{\circ}$ below the horizontal. $\mathbf{P}=2600 \mathrm{~N}$ - $53.5^{\circ}$

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## PROBLEM 2.12

For the hook support shown, determine by trigonometry the magnitude and direction of the resultant of the two forces applied to the support.

## SOLUTION



Using the law of cosines:

$$
\begin{aligned}
R^{2}= & (200 \mathrm{lb})^{2}+(300 \mathrm{lb})^{2} \\
& -2(200 \mathrm{lb})(300 \mathrm{lb}) \cos \left(45+65^{\circ}\right) \\
R= & 413.57 \mathrm{lb}
\end{aligned}
$$

Using the law of sines:

$$
\begin{aligned}
& \frac{\sin \alpha}{300 \mathrm{lb}}=\frac{\sin \left(45+65^{\circ}\right)}{413.57 \mathrm{lb}} \\
& \quad \alpha=42.972^{\circ} \\
& \beta=90+25-42.972^{\circ} \quad \mathbf{R}=414 \mathrm{lb} \text { ०. }_{\circ} 72.0^{\circ}
\end{aligned}
$$

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## SOLUTION



$$
\begin{aligned}
\tan \alpha & =\frac{8}{10} \\
\alpha & =38.66^{\circ} \\
\tan \beta & =\frac{6}{10} \\
\beta & =30.96^{\circ}
\end{aligned}
$$



Using the triangle rule:

$$
\begin{aligned}
38.66^{\circ}+30.96^{\circ}+\psi & =180^{\circ} \\
\psi & =110.38^{\circ}
\end{aligned}
$$

Using the law of cosines:

$$
\begin{aligned}
R^{2} & =(120 \mathrm{lb})^{2}+(40 \mathrm{lb})^{2}-2(120 \mathrm{lb})(40 \mathrm{lb}) \cos 110.38^{\circ} \\
R & =139.08 \mathrm{lb}
\end{aligned}
$$

Using the law of sines:

$$
\begin{aligned}
\frac{\sin \gamma}{40 \mathrm{lb}} & =\frac{\sin 110.38^{\circ}}{139.08 \mathrm{lb}} \\
\gamma & =15.64^{\circ} \\
\phi & =\left(90^{\circ}-\alpha\right)+\gamma \\
\phi & =\left(90^{\circ}-38.66^{\circ}\right)+15.64^{\circ} \\
\phi & =66.98^{\circ} \quad \mathbf{R}=139.1 \mathrm{lb} \quad \bar{\gamma} 67.0^{\circ}
\end{aligned}
$$



## SOLUTION

Using the force triangle and the laws of cosines and sines:
We have:

$$
\begin{aligned}
\gamma & =180^{\circ}-\left(50^{\circ}+25^{\circ}\right) \\
& =105^{\circ}
\end{aligned}
$$



Then

$$
\begin{aligned}
R^{2} & =(4 \mathrm{kips})^{2}+(6 \mathrm{kips})^{2}-2(4 \mathrm{kips})(6 \mathrm{kips}) \cos 105^{\circ} \\
& =64.423 \mathrm{kips}^{2} \\
R & =8.0264 \mathrm{kips}
\end{aligned}
$$

And

$$
\begin{aligned}
\frac{4 \mathrm{kips}}{\sin \left(25^{\circ}+\alpha\right)} & =\frac{8.0264 \mathrm{kips}}{\sin 105^{\circ}} \\
\sin \left(25^{\circ}+\alpha\right) & =0.48137 \\
25^{\circ}+\alpha & =28.775^{\circ} \\
\alpha & =3.775^{\circ}
\end{aligned}
$$

$$
\mathbf{R}=8.03 \mathrm{kips}>3.8^{\circ}
$$



## SOLUTION



The smallest force $P$ will be perpendicular to $R$.
(a) $\quad P=(50 \mathrm{~N}) \sin 25^{\circ}$
(b) $\quad R=(50 \mathrm{~N}) \cos 25^{\circ}$

$$
\begin{gathered}
\mathbf{P}=21.1 \mathrm{~N} \\
R=45.3 \mathrm{~N}
\end{gathered}
$$



## PROBLEM 2.16

Determine the $x$ and $y$ components of each of the forces shown.

## SOLUTION

Compute the following distances:

| $O A$ | $=\sqrt{600)^{2}+(800)^{2}}$ |
| ---: | :--- |
|  | $=1000 \mathrm{~mm}$ |
| $O B$ | $=\left(\sqrt{560)^{2}+(900)^{2}}\right.$ |
|  | $=1060 \mathrm{~mm}$ |
| $O C$ | $=(480)^{2}+(900)^{2}$ |
|  | $=1020 \mathrm{~mm}$ |



800-N Force:

$$
\begin{aligned}
& F_{x}=+(800 \mathrm{~N}) \frac{800}{1000} \\
& F_{y}=+(800 \mathrm{~N}) \frac{600}{1000}
\end{aligned}
$$

$$
F_{x}=+640 \mathrm{~N}
$$

$$
F_{y}=+480 \mathrm{~N}
$$

424-N Force:

$$
F_{x}=-(424 \mathrm{~N}) \frac{560}{1060}
$$

$$
F_{x}=-224 \mathrm{~N}
$$

$$
F_{y}=-(424 \mathrm{~N}) \frac{900}{1060}
$$

$$
F_{y}=-360 \mathrm{~N}
$$

408-N Force:

$$
\begin{aligned}
& F_{x}=+(408 \mathrm{~N}) \frac{480}{1020} \\
& F_{y}=-(408 \mathrm{~N}) \frac{900}{1020}
\end{aligned}
$$

$$
\begin{gathered}
F_{x}=+192.0 \mathrm{~N} \\
F_{y}=-360 \mathrm{~N}
\end{gathered}
$$

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## PROBLEM 2.17

Determine the $x$ and $y$ components of each of the forces shown.

## SOLUTION

Compute the following distances:

29-lb Force:

$$
\begin{aligned}
O A & =\left(\sqrt{84)^{2}+(80)^{2}}\right. \\
& =116 \mathrm{in} . \\
O B & =\left(\sqrt{28)^{2}+(96)^{2}}\right. \\
& =100 \mathrm{in} . \\
O C & =(48)^{2}+(90)^{2} \\
& =102 \mathrm{in} .
\end{aligned}
$$



$$
F_{x}=+21.0 \mathrm{lb}
$$

$$
F_{y}=+(29 \mathrm{lb}) \frac{80}{116}
$$

50-lb Force:

$$
F_{x}=+(29 \mathrm{lb}) \frac{84}{116}
$$

$$
F_{y}=+20.0 \mathrm{lb}
$$

$$
\begin{aligned}
& F_{x}=-(50 \mathrm{lb}) \frac{28}{100} \\
& F_{y}=+(50 \mathrm{lb}) \frac{96}{100}
\end{aligned}
$$

$$
F_{x}=-14.00 \mathrm{lb}
$$

$$
F_{y}=+48.0 \mathrm{lb}
$$

$$
F_{x}=+24.0 \mathrm{lb}
$$

$$
F_{y}=-(51 \mathrm{lb}) \frac{90}{102}
$$

$$
F_{y}=-45.0 \mathrm{lb}
$$

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## SOLUTION

| 40-lb Force: | $F_{x}=+(40 \mathrm{lb}) \cos 60^{\circ}$ | $F_{x}=20.0 \mathrm{lb}$ < |
| :---: | :---: | :---: |
|  | $F_{y}=-(40 \mathrm{lb}) \sin 60^{\circ}$ | $F_{y}=-34.6 \mathrm{lb}$ |
| 50-lb Force: | $F_{x}=-(50 \mathrm{lb}) \sin 50^{\circ}$ | $F_{x}=-38.3 \mathrm{lb}$ - |
|  | $F_{y}=-(50 \mathrm{lb}) \cos 50^{\circ}$ | $F_{y}=-32.1 \mathrm{lb}$ |
| 60-lb Force: | $F_{x}=+(60 \mathrm{lb}) \cos 25^{\circ}$ | $F_{x}=54.4 \mathrm{lb}$ |
|  | $F_{y}=+(60 \mathrm{lb}) \sin 25^{\circ}$ | $F_{y}=25.4 \mathrm{lb}$ < |

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## SOLUTION

80-N Force:
$F_{x}=+(80 \mathrm{~N}) \cos 40^{\circ}$

$$
\begin{aligned}
& F_{x}=61.3 \mathrm{~N} \\
& F_{y}=51.4 \mathrm{~N}
\end{aligned}
$$

$F_{y}=+(80 \mathrm{~N}) \sin 40^{\circ}$
120-N Force:
$F_{x}=+(120 \mathrm{~N}) \cos 70^{\circ}$
$F_{x}=41.0 \mathrm{~N}$
$F_{y}=+(120 \mathrm{~N}) \sin 70^{\circ}$
$F_{y}=112.8 \mathrm{~N}$
150-N Force:
$F_{x}=-(150 \mathrm{~N}) \cos 35^{\circ}$

$$
\begin{aligned}
& F_{x}=-122.9 \mathrm{~N} \\
& \qquad F_{y}=86.0 \mathrm{~N}
\end{aligned}
$$



## SOLUTION


(a)

$$
P \sin 35^{\circ}=300 \mathrm{lb}
$$

$$
P=\frac{300 \mathrm{lb}}{\sin 35^{\circ}}
$$

$$
P=523 \mathrm{lb}
$$

(b) Vertical component

$$
\begin{aligned}
P_{v} & =P \cos 35^{\circ} \\
& =(523 \mathrm{lb}) \cos 35^{\circ}
\end{aligned} P_{v}=428 \mathrm{lb}
$$



## SOLUTION

(a)

$$
\begin{aligned}
B C & =(\$ 50 \mathrm{~mm})^{2}+(720 \mathrm{~mm})^{2} \\
& =970 \mathrm{~mm}
\end{aligned}
$$

$$
P_{x}=P^{( }\left(\frac{650}{970}\right)
$$

or

$$
\begin{aligned}
P & =P_{x}\left(\frac{970}{650}\right) \\
& =325 \mathrm{~N}\left(\frac{970}{650}\right) \\
& =485 \mathrm{~N}
\end{aligned}
$$



$$
P=485 \mathrm{~N}
$$

(b)

$$
\begin{aligned}
P_{y} & =P\left(\frac{720}{970}\right) \\
& =485 \mathrm{~N}\left(\frac{720}{970}\right) \\
& =360 \mathrm{~N}
\end{aligned}
$$

$$
P_{y}=970 \mathrm{~N}
$$



## SOLUTION

(a)
(b)


$$
\begin{aligned}
& =\frac{350 \mathrm{lb}}{\cos 55^{\circ}} \\
& =610.21 \mathrm{lb}
\end{aligned}
$$

$$
P=610 \mathrm{lb} \text { 《 }
$$

$$
\begin{aligned}
P_{x} & =P \sin 55^{\circ} \\
& =(610.21 \mathrm{lb}) \sin 55^{\circ} \\
& =499.85 \mathrm{lb}
\end{aligned}
$$



## SOLUTION


(a)

$$
750 \mathrm{~N}=P \sin 20^{\circ}
$$

$$
P=2192.9 \mathrm{~N}
$$

$$
P=2190 \mathrm{~N}
$$

(b)

$$
\begin{aligned}
P_{A B C} & =P \cos 20^{\circ} \\
& =(2192.9 \mathrm{~N}) \cos 20^{\circ} \quad P_{A B C}=2060 \mathrm{~N}
\end{aligned}
$$



## PROBLEM 2.24

Determine the resultant of the three forces of Problem 2.16.
PROBLEM 2.16 Determine the $x$ and $y$ components of each of the forces shown.

## SOLUTION

Components of the forces were determined in Problem 2.16:

| Force | $x$ Comp. (N) | $y$ Comp. (N) |
| :---: | :---: | :---: |
| 800 lb | +640 | +480 |
| 424 lb | -224 | -360 |
| 408 lb | +192 | -360 |

$$
\begin{aligned}
\mathbf{R} & =R_{x} \mathbf{i}+R_{y} \mathbf{j} \\
& =(608 \mathrm{lb}) \mathbf{i}+(-240 \mathrm{lb}) \mathbf{j} \\
\tan \alpha & =\frac{R_{y}}{R_{x}} \\
& =\frac{240}{608} \\
\alpha & =21.541^{\circ}
\end{aligned}
$$



$$
R=\frac{240 \mathrm{~N}}{\sin \left(21.541^{\circ}\right)}
$$

$$
=653.65 \mathrm{~N}
$$

$\mathbf{R}=654 \mathrm{~N}$ © $21.5^{\circ}$

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## PROBLEM 2.25

Determine the resultant of the three forces of Problem 2.17.
PROBLEM 2.17 Determine the $x$ and $y$ components of each of the forces shown.

## SOLUTION

Components of the forces were determined in Problem 2.17:

| Force | $x$ Comp. (lb) | $y$ Comp. (lb) |
| :---: | :---: | :---: |
| 29 lb | +21.0 | +20.0 |
| 50 lb | -14.00 | +48.0 |
| 51 lb | +24.0 | -45.0 |

$$
\begin{array}{rlrl}
\mathbf{R} & =R_{x} \mathbf{i}+R_{y} \mathbf{j} \\
& =(31.0 \mathrm{lb}) \mathbf{i}+(23.0 \mathrm{lb}) \mathbf{j} \\
\tan \alpha & =\frac{R_{y}}{R_{x}} & R_{y}=23.0 \vec{j} \\
& =\frac{23.0}{31.0} \\
\alpha & =36.573^{\circ} \\
R & =\frac{23.0 \mathrm{lb}}{\sin \left(36.573^{\circ}\right)} & \\
& =38.601 \mathrm{lb} & \mathbf{R}=38.6 \mathrm{lb} \mathbb{R} 36.6^{\circ}
\end{array}
$$

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## PROBLEM 2.26

Determine the resultant of the three forces of Problem 2.18.

PROBLEM 2.18 Determine the $x$ and $y$ components of each of the forces shown.

## SOLUTION

| Force | $x$ Comp. (lb) | $y$ Comp. (lb) |
| :---: | :---: | :---: |
| 40 lb | +20.00 | -34.64 |
| 50 lb | -38.30 | -32.14 |
| 60 lb | +54.38 | +25.36 |
|  | $R_{x}=+36.08$ | $R_{y}=-41.42$ |

$$
\mathbf{R}=R_{x} \mathbf{i}+R_{y} \mathbf{j}
$$

$$
=(+36.08 \mathrm{lb}) \mathbf{i}+(-41.42 \mathrm{lb}) \mathbf{j}
$$

$$
\tan \alpha=\frac{R_{y}}{R_{x}}
$$

$$
\tan \alpha=\frac{41.42 \mathrm{lb}}{36.08 \mathrm{lb}}
$$

$$
\tan \alpha=1.14800
$$

$$
\alpha=48.942^{\circ}
$$

$$
R=\frac{41.42 \mathrm{lb}}{\sin 48.942^{\circ}}
$$



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## SOLUTION

Components of the forces were determined in Problem 2.19:

| Force | $x$ Comp. (N) | $y$ Comp. (N) |
| :---: | :---: | :---: |
| 80 N | +61.3 | +51.4 |
| 120 N | +41.0 | +112.8 |
| 150 N | -122.9 | +86.0 |

$$
\begin{array}{rlrl}
\mathbf{R} & =R_{x} \mathbf{i}+R_{y} \mathbf{j} \\
& =(-20.6 \mathrm{~N}) \mathbf{i}+(250.2 \mathrm{~N}) \mathbf{j} & & R \\
\tan \alpha & =\frac{R_{y}}{R_{x}} & & \underline{R}_{y}=250.2 \underline{j} \\
\tan \alpha & =\frac{250.2 \mathrm{~N}}{20.6 \mathrm{~N}} & & \underline{R}_{x}=-20.6 \underline{c^{\prime}} \\
\tan \alpha & =12.1456 \\
\alpha & =85.293^{\circ} & \mathbf{R}=251 \mathrm{~N} \triangle 85.3^{\circ} \\
R & =\frac{250.2 \mathrm{~N}}{\sin 85.293^{\circ}} & &
\end{array}
$$



## PROBLEM 2.28

For the collar loaded as shown, determine (a) the required value of $\alpha$ if the resultant of the three forces shown is to be vertical, (b) the corresponding magnitude of the resultant.

## SOLUTION

$$
\begin{align*}
R_{x} & =\Sigma F_{x} \\
& =(100 \mathrm{~N}) \cos \alpha+(150 \mathrm{~N}) \cos \left(\alpha+30^{\circ}\right)-(200 \mathrm{~N}) \cos \alpha \\
R_{x} & =-(100 \mathrm{~N}) \cos \alpha+(150 \mathrm{~N}) \cos \left(\alpha+30^{\circ}\right)  \tag{1}\\
R_{y} & =\Sigma F_{y} \\
& =-(100 \mathrm{~N}) \sin \alpha-(150 \mathrm{~N}) \sin \left(\alpha+30^{\circ}\right)-(200 \mathrm{~N}) \sin \alpha \\
R_{y} & =-(300 \mathrm{~N}) \sin \alpha-(150 \mathrm{~N}) \sin \left(\alpha+30^{\circ}\right) \tag{2}
\end{align*}
$$

(a) For $\mathbf{R}$ to be vertical, we must have $R_{x}=0$. We make $R_{x}=0$ in Eq. (1):

$$
\begin{aligned}
-100 \cos \alpha+150 \cos \left(\alpha+30^{\circ}\right) & =0 \\
-100 \cos \alpha+150\left(\cos \alpha \cos 30^{\circ}-\sin \alpha \sin 30^{\circ}\right) & =0 \\
29.904 \cos \alpha & =75 \sin \alpha \\
\tan \alpha & =\frac{29.904}{75} \\
& =0.39872 \\
\alpha & =21.738^{\circ}
\end{aligned}
$$

$$
\alpha=21.7^{\circ}
$$

(b) Substituting for $\alpha$ in Eq. (2):

$$
\begin{aligned}
R_{y} & =-300 \sin 21.738^{\circ}-150 \sin 51.738^{\circ} \\
& =-228.89 \mathrm{~N}
\end{aligned}
$$

$$
R=\left|R_{y}\right|=228.89 \mathrm{~N} \quad R=229 \mathrm{~N}
$$

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## PROBLEM 2.29

A hoist trolley is subjected to the three forces shown. Knowing that $\alpha=40^{\circ}$, determine (a) the required magnitude of the force $\mathbf{P}$ if the resultant of the three forces is to be vertical, (b) the corresponding magnitude of the resultant.

## SOLUTION

$$
\begin{align*}
& R_{x}=+\Sigma F_{x}=P+(200 \mathrm{lb}) \sin 40^{\circ}-(400 \mathrm{lb}) \cos 40^{\circ} \\
& R_{x}=P-177.860 \mathrm{lb}  \tag{1}\\
& R_{y}=+\Sigma F_{y}=(200 \mathrm{lb}) \cos 40^{\circ}+(400 \mathrm{lb}) \sin 40^{\circ} \\
& R_{y}=410.32 \mathrm{lb} \tag{2}
\end{align*}
$$

(a) For $\mathbf{R}$ to be vertical, we must have $R_{x}=0$.

Set

$$
\begin{aligned}
& R_{x}=0 \text { in Eq. } \\
& 0=P-177.860 \mathrm{lb}
\end{aligned}
$$

$$
P=177.860 \mathrm{lb} \quad P=177.9 \mathrm{lb}
$$

(b) Since $\mathbf{R}$ is to be vertical:

$$
R=R_{y}=410 \mathrm{lb} \quad R=410 \mathrm{lb}
$$



## PROBLEM 2.30

A hoist trolley is subjected to the three forces shown. Knowing that $P=250 \mathrm{lb}$, determine (a) the required value of $\alpha$ if the resultant of the three forces is to be vertical, $(b)$ the corresponding magnitude of the resultant.

## SOLUTION

$$
\begin{align*}
& R_{x}=\xrightarrow{+} \Sigma F_{x}=250 \mathrm{lb}+(200 \mathrm{lb}) \sin \alpha-(400 \mathrm{lb}) \cos \alpha \\
& R_{x}=250 \mathrm{lb}+(200 \mathrm{lb}) \sin \alpha-(400 \mathrm{lb}) \cos \alpha \tag{1}
\end{align*}
$$

(a) For $\mathbf{R}$ to be vertical, we must have $R_{x}=0$.

Set

$$
\begin{aligned}
R_{x} & =0 \text { in Eq. (1) } \\
0 & =250 \mathrm{lb}+(200 \mathrm{lb}) \sin \alpha-(400 \mathrm{lb}) \cos \alpha
\end{aligned}
$$

$$
(400 \mathrm{lb}) \cos \alpha=(200 \mathrm{lb}) \sin \alpha+250 \mathrm{lb}
$$

$$
2 \cos \alpha=\sin \alpha+1.25
$$

$$
4 \cos ^{2} \alpha=\sin ^{2} \alpha+2.5 \sin \alpha+1.5625
$$

$$
4\left(1-\sin ^{2} \alpha\right)=\sin ^{2} \alpha+2.5 \sin \alpha+1.5625
$$

$$
0=5 \sin ^{2} \alpha+2.5 \sin \alpha-2.4375
$$

Using the quadratic formula to solve for the roots gives

$$
\sin \alpha=0.49162
$$

or $\quad \alpha=29.447^{\circ} \quad \alpha=29.4^{\circ}$
(b) Since $\mathbf{R}$ is to be vertical:

$$
R=R_{y}=(200 \mathrm{lb}) \cos 29.447^{\circ}+(400 \mathrm{lb}) \sin 29.447^{\circ} \quad \mathbf{R}=371 \mathrm{lb}
$$



## PROBLEM 2.31

For the post loaded as shown, determine $(a)$ the required tension in rope $A C$ if the resultant of the three forces exerted at point $C$ is to be horizontal, (b) the corresponding magnitude of the resultant.

## SOLUTION

$$
\begin{align*}
& R_{x}=\Sigma F_{x}^{=}-\frac{960}{1460}{ }_{A C}+\frac{24}{25}(500 \mathrm{~N})+\frac{4}{5}(200 \mathrm{~N}) \\
& R_{x}=-{\frac{48}{73} T_{A C}+640 \mathrm{~N}}_{R_{y}=\Sigma F_{y}-\frac{1100}{T}_{1460}^{A C}}+\frac{7}{25}(500 \mathrm{~N})-\frac{3}{5}(200 \mathrm{~N})  \tag{1}\\
& R_{y}=-\frac{55}{73} T_{A C}+20 \mathrm{~N}
\end{align*}
$$

(a) For $\mathbf{R}$ to be horizontal, we must have $R_{y}=0$.

Set $R=0$ in Eq. (2):

$$
-\frac{55}{73} T_{A C}+20 \mathrm{~N}=0
$$

$$
T_{A C}=26.545 \mathrm{~N}
$$

$$
T_{A C}=26.5 \mathrm{~N}
$$

(b) Substituting for $T_{A C}$ into Eq. (1) gives

$$
\begin{aligned}
R_{x} & =-\frac{48}{73}(26.545 \mathrm{~N})+640 \mathrm{~N} \\
R_{x} & =622.55 \mathrm{~N} \\
R & =R_{x}=623 \mathrm{~N}
\end{aligned}
$$

$$
R=623 \mathrm{~N}
$$



## SOLUTION

## Free-Body Diagram

## Force Triangle



Law of sines:

$$
\frac{T_{A C}}{\sin 60^{\circ}}=\frac{T_{B C}}{\sin 35^{\circ}}=\frac{6 \mathrm{kN}}{\sin 85^{\circ}}
$$

(a)
(b)

$$
\begin{aligned}
T_{A C}=\frac{6 \mathrm{kN}}{\sin 85^{\circ}}\left(\sin 60^{\circ}\right) & T_{A C}=5.22 \mathrm{kN} \triangleleft \\
T_{B C}=\frac{6 \mathrm{kN}}{\sin 85^{\circ}}\left(\sin 35^{\circ}\right) & T_{B C}=3.45 \mathrm{kN} \triangleleft
\end{aligned}
$$

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## SOLUTION

Free-Body Diagram


## Force Triangle



Law of sines:
$\frac{T_{A C}}{\sin 60^{\circ}}=\frac{T_{B C}}{\sin 40^{\circ}}=\frac{400 \mathrm{lb}}{\sin 80^{\circ}}$
(a)
(b)

$$
\begin{array}{ll}
T_{A C}=\frac{400 \mathrm{lb}}{\sin 80^{\circ}}\left(\sin 60^{\circ}\right) & T_{A C}=352 \mathrm{lb} \\
T_{B C}=\frac{400 \mathrm{lb}}{\sin 80^{\circ}}\left(\sin 40^{\circ}\right) & T_{B C}=261 \mathrm{lb}
\end{array}
$$

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## PROBLEM 2.34

Two cables are tied together at $C$ and are loaded as shown. Determine the tension $(a)$ in cable $A C,(b)$ in cable $B C$.

## SOLUTION

## Free-Body Diagram

Force Triangle


$$
\begin{aligned}
W & =\mathrm{mg} \\
& =(200 \mathrm{~kg})\left(9.81 \mathrm{~m} / \mathrm{s}^{2}\right) \\
& =1962 \mathrm{~N}
\end{aligned}
$$



Law of sines:

$$
\frac{T_{A C}}{\sin 15^{\circ}}=\frac{T_{B C}}{\sin 105^{\circ}}=\frac{1962 \mathrm{~N}}{\sin 60^{\circ}}
$$

(a)
(b)

$$
T_{A C}=\frac{(1962 \mathrm{~N}) \sin 15^{\circ}}{\sin 60^{\circ}}
$$

$$
T_{A C}=586 \mathrm{~N}
$$

$$
T_{B C}=\frac{(1962 \mathrm{~N}) \sin 105^{\circ}}{\sin 60^{\circ}}
$$

$$
T_{B C}=2190 \mathrm{~N} \hookrightarrow
$$



## SOLUTION

## Free Body Diagram at $C$ :

$$
\begin{gathered}
\Sigma \mathbf{F}_{x}=0: \quad-\frac{12 \mathrm{ft}}{12.5 \mathrm{ft}} T_{A C}+\frac{7.5 \mathrm{ft}}{8.5 \mathrm{ft}} T_{B C}=0 \\
T_{B C}=1.08800 T_{A C} \\
\Sigma \mathbf{F}_{y}=0: \quad \frac{3.5 \mathrm{ft}}{12 \mathrm{ft}} T_{A C}+\frac{4 \mathrm{ft}}{8.5 \mathrm{ft}} T_{B C}-396 \mathrm{lb}=0 \\
\frac{3.5 \mathrm{ft}}{12.5 \mathrm{ft}} T_{A C}+\frac{4 \mathrm{ft}}{8.5 \mathrm{ft}}\left(1.088000 T_{A C}\right)-396 \mathrm{lb}=0 \\
(0.28000+0.51200) T_{A C}=396 \mathrm{lb}
\end{gathered}
$$

(a)

$$
T_{A C}=500.0 \mathrm{lb}
$$



$$
T_{A C}=500 \mathrm{lb}
$$

(b)

$$
T_{B C}=(1.08800)(500.0 \mathrm{lb})
$$

b)

$$
T_{B C}=544 \mathrm{lb}
$$



## SOLUTION

## Free-Body Diagram



Law of sines:

$$
\frac{T_{A C}}{\sin 35^{\circ}}=\frac{T_{B C}}{\sin 75^{\circ}}=\frac{500 \mathrm{~N}}{\sin 70^{\circ}}
$$

(a)
(b)

$$
T_{A C}=\frac{500 \mathrm{~N}}{\sin 70^{\circ}} \sin 35^{\circ}
$$

$$
T_{A C}=305 \mathrm{~N}
$$

$$
T_{B C}=\frac{500 \mathrm{~N}}{\sin 70^{\circ}} \sin 75^{\circ}
$$

$$
T_{B C}=514 \mathrm{~N}
$$

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$$
\begin{aligned}
& T_{C}=5.87 \mathrm{kips} \measuredangle \\
& T_{D}=9.14 \mathrm{kips}
\end{aligned}
$$

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## SOLUTION

Substituting for $T_{D}$ into Eq. (1) gives:

$$
\begin{aligned}
T_{B}-6 \mathrm{kips}-(14.0015 \mathrm{kips}) \cos 40^{\circ} & =0 \\
T_{B} & =16.7258 \mathrm{kips}
\end{aligned}
$$

$$
T_{B}=16.73 \mathrm{kips}
$$

$$
T_{D}=14.00 \mathrm{kips}
$$

$$
\begin{aligned}
& \xrightarrow{+} \Sigma F_{x}=0 \\
& \dagger \Sigma F_{y}=0 \\
& T_{B}-6 \mathrm{kips}-T_{D} \cos 40^{\circ}=0 \\
& T_{D} \sin 40^{\circ}-9 \text { kips }=0 \\
& T_{D}=\frac{9 \mathrm{kips}}{\sin 40^{\circ}} \\
& T_{D}=14.0015 \mathrm{kips}
\end{aligned}
$$



## SOLUTION

## Free-Body Diagram


$\xrightarrow{+} \Sigma F_{x}=0-T_{C A} \sin 30 \cdot+T_{C B} \sin 30 \cdot-P \cos 45^{\circ}-200 \mathrm{~N}=0$
For $P=200 \mathrm{~N}$ we have,

$$
-0.5 T_{C A}+0.5 T_{C B}+212.13-200=0
$$

$+\dagger \Sigma F_{y}=0$
$T_{C A} \cos 30^{\circ}-T_{C B} \cos 30 \cdot-P \sin 45=0$

$$
0.86603 T_{C A}+0.86603 T_{C B}-212.13=0
$$

Solving equations (1) and (2) simultaneously gives,

$$
\begin{gathered}
T_{C A}=134.6 \mathrm{~N} \\
T_{C B}=110.4 \mathrm{~N}
\end{gathered}
$$



## SOLUTION

Free-Body Diagram
Resolving the forces into $x$ - and $y$-directions:

$$
\mathbf{R}=\mathbf{P}+\mathbf{Q}+\mathbf{F}_{A}+\mathbf{F}_{B}=0
$$

Substituting components:

$$
\begin{aligned}
\mathbf{R}= & -(500 \mathrm{lb}) \mathbf{j}+\left[(650 \mathrm{lb}) \cos 50^{\circ}\right] \mathbf{i} \\
& -\left[(650 \mathrm{lb}) \sin 50^{\circ}\right] \mathbf{j} \\
& +F_{B} \mathbf{i}-\left(F_{A} \cos 50^{\circ}\right) \mathbf{i}+\left(F_{A} \sin 50^{\circ}\right) \mathbf{j}=0
\end{aligned}
$$

In the $y$-direction (one unknown force):

$$
-500 \mathrm{lb}-(650 \mathrm{lb}) \sin 50^{\circ}+F_{A} \sin 50^{\circ}=0
$$



Thus,

$$
\begin{aligned}
F_{A} & =\frac{500 \mathrm{lb}+(650 \mathrm{lb}) \sin 50^{\circ}}{\sin 50^{\circ}} \\
& =1302.70 \mathrm{lb}
\end{aligned}
$$

$$
F_{A}=1303 \mathrm{lb}
$$

In the $x$-direction:
$(650 \mathrm{lb}) \cos 50^{\circ}+F_{B}-F_{A} \cos 50^{\circ}=0$
Thus,

$$
\begin{aligned}
F_{B} & =F_{A} \cos 50^{\circ}-(650 \mathrm{lb}) \cos 50^{\circ} \\
& =(1302.70 \mathrm{lb}) \cos 50^{\circ}-(650 \mathrm{lb}) \cos 50^{\circ} \\
& =419.55 \mathrm{lb}
\end{aligned}
$$

$$
F_{B}=420 \mathrm{lb}
$$

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## SOLUTION

$$
\begin{align*}
& +5 F_{x}=0: T_{A C B} \cos 10^{\circ}-T_{A C B} \cos 30^{\circ}-T_{C D} \cos 30^{\circ}=0 \\
& +T_{C D}=0.137158 T_{A C B} \\
& +5 F_{y}=0: T_{A C B} \sin 10^{\circ}+T_{A C B} \sin 30^{\circ}+T_{C D} \sin 30^{\circ}-200=0 \\
& 0.67365 T_{A C B}+0.5 T_{C D}=200 \tag{1}
\end{align*}
$$

(a) Substitute (1) into (2): $0.67365 T_{A C B}+0.5\left(0.137158 T_{A C B}\right)=200$

$$
T_{A C B}=269.46 \mathrm{lb}
$$

$$
T_{A C B}=269 \mathrm{lb}
$$

(b) From (1):

$$
T_{C D}=0.137158(269.46 \mathrm{lb})
$$

$$
T_{C D}=37.0 \mathrm{lb}
$$

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## SOLUTION


(a) $\quad W=216 \mathrm{lb}$
(b) $T_{A C B}=304 \mathrm{lb}$

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## PROBLEM 2.43

For the cables of prob. 2.32, find the value of $\alpha$ for which the tension is as small as possible $(a)$ in cable $b c,(b)$ in both cables simultaneously. In each case determine the tension in each cable.

## SOLUTION

## Free-Body Diagram

## Force Triangle


(a) For a minimum tension in cable $B C$, set angle between cables to 90 degrees.

By inspection,

$$
\begin{aligned}
& T_{A C}=(6 \mathrm{kN}) \cos 35 \\
& T_{B C}=(6 \mathrm{kN}) \sin 35
\end{aligned}
$$

$$
T_{A C}=4.91 \mathrm{kN}
$$

$$
T_{B C}=3.44 \mathrm{kN}
$$

(b) For equal tension in both cables, the force triangle will be an isosceles.

Therefore, by inspection,


$$
T_{A C}=T_{B C}=(1 / 2) \frac{6 \mathrm{kN}}{\cos 35^{\circ}} \quad T_{A C}=T_{B C}=3.66 \mathrm{kN}
$$

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## SOLUTION

## Free-Body Diagram



## Force Triangle


(a) Law of cosines

$$
P^{2}=(600)^{2}+(750)^{2}-2(600)(750) \cos \left(25^{\circ}+45^{\circ}\right)
$$

$$
P=784.02 \mathrm{~N} \quad P=784 \mathrm{~N}
$$

(b) Law of sines

$$
\begin{aligned}
\frac{\sin \beta}{600 \mathrm{~N}} & =\frac{\sin \left(25^{\circ}+45^{\circ}\right)}{784.02 \mathrm{~N}} \\
\beta & =46.0^{\circ} \quad \therefore \alpha=46.0^{\circ}+25^{\circ} \quad \alpha=71.0^{\circ}
\end{aligned}
$$

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## SOLUTION

Free-Body Diagram: $C$


Force triangle is isosceles with
(a) $\quad P=2(800 \mathrm{~N}) \cos 47.5^{\circ}=1081 \mathrm{~N}$

Since $P>0$, the solution is correct.
(b)
$\alpha=180^{\circ}-50^{\circ}-47.5^{\circ}=82.5^{\circ}$
$P=1081 \mathrm{~N}$ 4
$2 \beta=180^{\circ}-85^{\circ}$
$\beta=47.5^{\circ}$
$\alpha=82.5^{\circ}$ 4

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## SOLUTION

## Free-Body Diagram



$$
\begin{aligned}
P^{2} & =(1200 \mathrm{~N})^{2}+(600 \mathrm{~N})^{2}-2(1200 \mathrm{~N})(600 \mathrm{~N}) \cos 85^{\circ} \\
P & =1294 \mathrm{~N}
\end{aligned}
$$

Since $P .1200 \mathrm{~N}$, the solution is correct.

$$
P=1294 \mathrm{~N} \text { 4 }
$$

(b) Law of sines:

$$
\begin{aligned}
\frac{\sin \beta}{1200 \mathrm{~N}} & =\frac{\sin 85^{\circ}}{1294 \mathrm{~N}} \\
\beta & =67.5^{\circ} \\
\alpha & =180^{\circ}-50^{\circ}-67.5^{\circ}
\end{aligned} \quad \alpha=62.5^{\circ}
$$

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## SOLUTION



$$
\Sigma F_{x}=0:-T_{B C}-Q \cos 60^{\circ}+75 \mathrm{lb}=0
$$

$T_{B C}=75 \mathrm{lb}-Q \cos 60^{\circ}$
$\Sigma F_{y}=0: T_{A C}-Q \sin 60^{\circ}=0$
$T_{A C}=Q \sin 60^{\circ}$
Requirement: $\quad T_{A C}=60 \mathrm{lb}$
From Eq. (2): $\quad Q \sin 60^{\circ}=60 \mathrm{lb}$

$$
Q=69.3 \mathrm{lb}
$$

Requirement: $\quad T_{B C}=60 \mathrm{lb}:$
From Eq. (1): $75 \mathrm{lb}-Q \cos 60^{\circ}=60 \mathrm{lb}$

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## SOLUTION

(a) Free Body: Collar A


## Force Triangle

$$
\frac{P}{4.5}=\frac{50 \mathrm{lb}}{20.5} \quad P=10.98 \mathrm{lb}
$$

## Force Triangle

$$
\frac{P}{15}=\frac{50 \mathrm{lb}}{25} \quad P=30.0 \mathrm{lb}
$$



## PROBLEM 2.49

Collar $A$ is connected as shown to a $50-\mathrm{lb}$ load and can slide on a frictionless horizontal rod. Determine the distance $x$ for which the collar is in equilibrium when $P=48 \mathrm{lb}$.

## SOLUTION

## Free Body: Collar A



## Force Triangle



$$
\begin{aligned}
N^{2} & =(50)^{2}-(48)^{2}=196 \\
N & =14.00 \mathrm{lb}
\end{aligned}
$$

## Similar Triangles

$$
\frac{x}{20 \mathrm{in} .}=\frac{48 \mathrm{lb}}{14 \mathrm{lb}}
$$



$$
x=68.6 \mathrm{in} .
$$



## PROBLEM 2.50

A movable bin and its contents have a combined weight of 2.8 kN . Determine the shortest chain sling $A C B$ that can be used to lift the loaded bin if the tension in the chain is not to exceed 5 kN .

## SOLUTION

## Free-Body Diagram



Isosceles Force Triangle


Law of sines: $\quad \sin \alpha=\frac{\frac{1}{2}(2.8 \mathrm{kN})}{T_{A C}}$

$$
\begin{gathered}
T_{A C}=5 \mathrm{kN} \\
\sin \alpha=\frac{\frac{1}{2}(2.8 \mathrm{kN})}{5 \mathrm{kN}}
\end{gathered}
$$

$$
\alpha=16.2602^{\circ}
$$

From Eq. (1): $\tan 16.2602^{\circ}=\frac{h}{0.6} \mathrm{~m} \quad \therefore h=0.175000 \mathrm{~m}$
Half-length of chain $=A C=(Q .6 \mathrm{~m})^{2}+(0.175 \mathrm{~m})^{2}$

$$
=0.625 \mathrm{~m}
$$

Total length:

$$
=2 \times 0.625 \mathrm{~m}
$$

$$
1.250 \mathrm{~m}
$$

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and-pulley arrangements as shown. Determine for each arrangement the tension in the rope. (Hint: The tension in the rope is the same on each side of a simple pulley. This can be
A 600-lb crate is supported by several rope-

(b)

(c)

(d)

(c)
proved by the methods of Ch. 4.)

## PROBLEM 2.51

 proved$\qquad$

## SOLUTION

## Free-Body Diagram of Pulley

(a)

$$
\begin{aligned}
+\dagger \Sigma F_{y}=0: 2 T-(600 \mathrm{lb}) & =0 \\
& T=\frac{1}{2}(600 \mathrm{lb})
\end{aligned}
$$



## SOLUTION

## Free-Body Diagram of Pulley and Crate

(b)


$$
\begin{aligned}
+\dagger \Sigma F_{y}=0: 3 T-(600 \mathrm{lb}) & =0 \\
T & =\frac{1}{3}(600 \mathrm{lb})
\end{aligned}
$$

$$
T=200 \mathrm{lb} \measuredangle
$$

(d)

$\begin{aligned}+\dagger \Sigma F_{y}=0: 4 T-(600 \mathrm{lb}) & =0 \\ & T={ }_{\frac{1}{4}}^{1}(600 \mathrm{lb})\end{aligned}$

$$
T=150.0 \mathrm{lb}
$$

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## SOLUTION

## Free-Body Diagram: Pulley A



$$
\begin{aligned}
\xrightarrow{+} \Sigma F_{x} & =0:-2 P\left(\frac{5}{\sqrt{28 I}}\right)+P \cos \alpha=0 \\
\cos \alpha & =0.59655 \\
\alpha & = \pm 53.377^{\circ}
\end{aligned}
$$

For $\alpha=+53.377^{\circ}$ :
$+\sum_{y}^{\dagger \Sigma F_{y}=0: 2 P\left(\frac{16)}{(\sqrt{281}}\right)} \sin 53.377^{\circ}-1962 \mathrm{~N}=0$

$$
\mathbf{P}=724 \mathrm{~N}<53.4^{\circ}
$$

For $\alpha=-53.377^{\circ}$ :
$+\dagger_{y} \Sigma F_{y}=0: 2 P^{(16)+P}(\sqrt{\sqrt{281}}) \sin \left(-53.377^{\circ}\right)-1962 \mathrm{~N}=0$

$$
\mathbf{P}=1773<53.4^{\circ}
$$

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## PROBLEM 2.54

A load $\mathbf{Q}$ is applied to the pulley $C$, which can roll on the cable $A C B$. The pulley is held in the position shown by a second cable $C A D$, which passes over the pulley $A$ and supports a load $\mathbf{P}$. Knowing that $P=750 \mathrm{~N}$, determine (a) the tension in cable $A C B,(b)$ the magnitude of $\operatorname{load} \mathbf{Q}$.

## SOLUTION

## Free-Body Diagram: Pulley C


(a) $\xrightarrow{+} \Sigma F_{x}=0: T_{A C B}\left(\cos 25^{\circ}-\cos 55^{\circ}\right)-(750 \mathrm{~N}) \cos 55^{\circ}=0$

Hence: $\quad T_{A C B}=1292.88 \mathrm{~N}$

$$
T_{A C B}=1293 \mathrm{~N}
$$

(b) $\quad+\dagger \Sigma F_{y}=0: T_{A C B}\left(\sin 25^{\circ}+\sin 55^{\circ}\right)+(750 \mathrm{~N}) \sin 55^{\circ}-Q=0$

$$
(1292.88 \mathrm{~N})\left(\sin 25^{\circ}+\sin 55^{\circ}\right)+(750 \mathrm{~N}) \sin 55^{\circ}-Q=0
$$

or

$$
Q=2219.8 \mathrm{~N} \quad Q=2220 \mathrm{~N}
$$

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## PROBLEM 2.55

An $1800-\mathrm{N}$ load $\mathbf{Q}$ is applied to the pulley $C$, which can roll on the cable $A C B$. The pulley is held in the position shown by a second cable $C A D$, which passes over the pulley $A$ and supports a load $\mathbf{P}$. Determine $(a)$ the tension in cable $A C B$, (b) the magnitude of load $\mathbf{P}$.

## SOLUTION

## Free-Body Diagram: Pulley C

$$
\xrightarrow{+} \Sigma F_{x}=0: T_{A C B}\left(\cos 25^{\circ}-\cos 55^{\circ}\right)-P \cos 55^{\circ}=0
$$


or

$$
P=0.58010 T_{A C B}(1)
$$

$$
+\dagger \Sigma F_{y}=0: T_{A C B}\left(\sin 25^{\circ}+\sin 55^{\circ}\right)+P \sin 55^{\circ}-1800 \mathrm{~N}=0
$$

or

$$
1.24177 T_{A C B}+0.81915 P=1800 \mathrm{~N}(2)
$$

(a) Substitute Equation (1) into Equation (2):

$$
1.24177 T_{A C B}+0.81915\left(0.58010 T_{A C B}\right)=1800 \mathrm{~N}
$$

Hence:

$$
T_{A C B}=1048.37 \mathrm{~N}
$$

$$
T_{A C B}=1048 \mathrm{~N} 4
$$

(b) $\quad \operatorname{Using}(1), \quad P=0.58010(1048.37 \mathrm{~N})=608.16 \mathrm{~N}$

$$
P=608 \mathrm{~N} \text { 4 }
$$

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## SOLUTION

$$
\begin{aligned}
F_{h} & =F \cos 65^{\circ} \\
& =(900 \mathrm{~N}) \cos 65^{\circ} \\
F_{h} & =380.36 \mathrm{~N}
\end{aligned}
$$


(a)
$F_{x}=F_{h} \sin 20^{\circ}$

$$
=(380.36 \mathrm{~N}) \sin 20^{\circ}
$$

$F_{x}=-130.091 \mathrm{~N}$,
$F_{x}=-130.1 \mathrm{~N}$
$F_{y}=F \sin 65^{\circ}$
$=(900 \mathrm{~N}) \sin 65^{\circ}$
$F_{y}=+815.68 \mathrm{~N}$,
$F_{z}=F_{h} \cos 20^{\circ}$

$$
=(380.36 \mathrm{~N}) \cos 20^{\circ}
$$

$$
F_{z}=+357.42 \mathrm{~N}
$$

$$
F_{z}=+357 \mathrm{~N}
$$

(b)

$$
\begin{array}{ll}
\cos \theta_{x}=\frac{F_{x}}{F}=\frac{-130.091 \mathrm{~N}}{900 \mathrm{~N}} & \theta_{x}=98.3^{\circ} . \\
\cos \theta_{y}=\frac{F_{y}}{F}=\frac{+815.68 \mathrm{~N}}{900 \mathrm{~N}} & \theta_{y}=25.0^{\circ} . \\
\cos \theta_{z}=\frac{F_{z}}{F}=\frac{+357.42 \mathrm{~N}}{900 \mathrm{~N}} & \theta_{z}=66.6^{\circ} .
\end{array}
$$



## SOLUTION

$$
\begin{aligned}
F_{h} & =F \sin 35^{\circ} \\
& =(750 \mathrm{~N}) \sin 35^{\circ} \\
F_{h} & =430.18 \mathrm{~N}
\end{aligned}
$$


(a)
$F_{x}=F_{h} \cos 25^{\circ}$

$$
=(430.18 \mathrm{~N}) \cos 25^{\circ}
$$

$F_{x}=+389.88 \mathrm{~N}$,

$$
F_{x}=+390 \mathrm{~N}
$$

$F_{y}=F \cos 35^{\circ}$ $=(750 \mathrm{~N}) \cos 35^{\circ}$
$F_{y}=+614.36 \mathrm{~N}$,
$F_{z}=F_{h} \sin 25^{\circ}$
$=(430.18 \mathrm{~N}) \sin 25^{\circ}$

$$
\begin{gathered}
F_{y}=+614 \mathrm{~N} \\
F_{z}=+181.8 \mathrm{~N}
\end{gathered}
$$

$F_{z}=+181.802 \mathrm{~N}$
$\cos \theta_{x}=\frac{\underline{F}_{x}}{F}=\frac{+389.88 \mathrm{~N}}{750 \mathrm{~N}}$

$$
\theta_{x}=58.7^{\circ}
$$

$$
\cos \theta_{y}=\frac{F_{y}}{F}=\frac{+614.36 \mathrm{~N}}{750 \mathrm{~N}}
$$

$$
\theta_{y}=35.0^{\circ}
$$

$$
\cos \theta_{z}=\frac{F_{z}}{F}=\frac{+181.802 \mathrm{~N}}{750 \mathrm{~N}}
$$

$$
\theta_{z}=76.0^{\circ}
$$

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## SOLUTION

(a)
$F_{x}=(120 \mathrm{lb}) \cos 60^{\circ} \cos 20^{\circ}$
$F_{x}=56.382 \mathrm{lb} \quad F_{x}=+56.4 \mathrm{lb}$
$F_{y}=-(120 \mathrm{lb}) \sin 60^{\circ}$
$F_{y}=-103.923 \mathrm{lb} \quad F_{y}=-103.9 \mathrm{lb}$
$F_{z}=-(120 \mathrm{lb}) \cos 60^{\circ} \sin 20^{\circ}$
$F_{z}=-20.521 \mathrm{lb}$
$\cos \theta_{x}=\frac{F_{x}}{F} \frac{56.382 \mathrm{lb}}{120 \mathrm{lb}}$

$$
F_{z}=-20.5 \mathrm{lb}
$$

(b)
$\cos \theta_{y}=\frac{F_{y}}{F}=\frac{-103.923 \mathrm{lb}}{120 \mathrm{lb}}$
$\theta_{y}=150.0^{\circ}$ 《
$\cos \theta_{z}=\frac{F_{z}}{F}=\frac{-20.52 \mathrm{lb}}{120 \mathrm{lb}}$
$\theta_{z}=99.8^{\circ}$

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## SOLUTION

(a)

$$
\begin{array}{rlrl}
F_{x} & =(85 \mathrm{lb}) \sin 36^{\circ} \sin 48^{\circ} & \\
& =37.129 \mathrm{lb} & & F_{x}=37.1 \mathrm{lb} \\
F_{y} & =-(85 \mathrm{lb}) \cos 36^{\circ} & & \\
& =-68.766 \mathrm{lb} & F_{y}=-68.8 \mathrm{lb} \\
F_{z} & =(85 \mathrm{lb}) \sin 36^{\circ} \cos 48^{\circ} & & \\
& =33.431 \mathrm{lb} & F_{z}=33.4 \mathrm{lb}
\end{array}
$$

(b)

$$
\begin{array}{ll}
\cos \theta_{x}=\frac{F_{\underline{x}}}{F}=\frac{37.129 \mathrm{lb}}{85 \mathrm{lb}} & \theta_{x}=64.1^{\circ} 4 \\
\cos \theta_{y}=\frac{F_{y}}{F}=\frac{-68.766 \mathrm{lb}}{85 \mathrm{lb}} & \theta_{y}=144.0^{\circ} \\
\cos \theta_{z}=\frac{F_{\underline{z}}}{F}=\frac{33.431 \mathrm{lb}}{85 \mathrm{lb}} & \theta_{z}=66.8^{\circ}
\end{array}
$$

## PROBLEM 2.60

A gun is aimed at a point $A$ located $35^{\circ}$ east of north. Knowing that the barrel of the gun forms an angle of $40^{\circ}$ with the horizontal and that the maximum recoil force is 400 N , determine (a) the $x, y$, and $z$ components of that force, $(b)$ the values of the angles $\theta_{x}, \theta_{y}$, and $\theta_{z}$ defining the direction of the recoil force. (Assume that the $x, y$, and $z$ axes are directed, respectively, east, up, and south.)

## SOLUTION

Recoil force

$$
\begin{gathered}
F=400 \mathrm{~N} \\
\therefore F_{H}=(400 \mathrm{~N}) \cos 40^{\circ} \\
=306.42 \mathrm{~N}
\end{gathered}
$$


(a)

$$
F_{y}=-257 \mathrm{~N} \text { 4 }
$$

$$
=+251.00 \mathrm{~N}
$$

(b)

$$
\begin{aligned}
F_{x} & =-F_{H} \sin 35^{\circ} \\
& =-(306.42 \mathrm{~N}) \sin 35^{\circ} \\
& =-175.755 \mathrm{~N} \quad F_{x}=-175.8 \mathrm{~N} 4
\end{aligned}
$$

$$
F_{y}=-F \sin 40^{\circ}
$$

$$
=-(400 \mathrm{~N}) \sin 40^{\circ}
$$

$$
=-257.12 \mathrm{~N}
$$

$$
F_{z}=+F_{H} \cos 35^{\circ}
$$

$$
=+(306.42 \mathrm{~N}) \cos 35^{\circ}
$$

$$
F_{z}=+251 \mathrm{~N} \text { 4 }
$$

$$
\begin{array}{ll}
\cos \theta_{x}=\frac{F_{x}}{F}=\frac{-175.755 \mathrm{~N}}{400 \mathrm{~N}} & \theta_{x=116.1^{\circ} \leftharpoonup}^{\cos \theta_{y}=\frac{F_{y}}{F}=\frac{-257.12 \mathrm{~N}}{400 \mathrm{~N}}} \\
\cos \theta_{z}=\frac{F_{z}}{F}=\frac{251.00 \mathrm{~N}}{400 \mathrm{~N}} & \theta_{y}=130.0^{\circ} \text { « }
\end{array}
$$

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## PROBLEM 2.61

Solve Problem 2.60, assuming that point $A$ is located $15^{\circ}$ north of west and that the barrel of the gun forms an angle of $25^{\circ}$ with the horizontal.

PROBLEM 2.60 A gun is aimed at a point $A$ located $35^{\circ}$ east of north. Knowing that the barrel of the gun forms an angle of $40^{\circ}$ with the horizontal and that the maximum recoil force is 400 N , determine (a) the $x, y$, and $z$ components of that force, $(b)$ the values of the angles $\theta_{x}, \theta_{y}$, and $\theta_{z}$ defining the direction of the recoil force. (Assume that the $x, y$, and $z$ axes are directed, respectively, east, up, and south.)

## SOLUTION

Recoil force

$$
F=400 \mathrm{~N}
$$


(a)

$$
F_{z}=+F_{H} \sin 15^{\circ}
$$

$$
=+(362.52 \mathrm{~N}) \sin 15^{\circ}
$$

$$
=+93.827 \mathrm{~N} \quad F_{z}=+93.8 \mathrm{~N}
$$

(b)

$$
\begin{array}{ll}
\cos \theta_{x}=\frac{F_{x}}{F}=\frac{+350.17 \mathrm{~N}}{400 \mathrm{~N}} & \theta_{x}=28.9^{\circ} \leftharpoonup \\
\cos \theta_{y}=\frac{F_{y}}{F}=\frac{-169.047 \mathrm{~N}}{400 \mathrm{~N}} & \theta_{y}=115.0^{\circ} 4 \\
\cos \theta_{z}=\frac{F_{z}}{F}=\frac{+93.827 \mathrm{~N}}{400 \mathrm{~N}} & \theta_{z}=76.4^{\circ}<
\end{array}
$$

$$
\begin{array}{rlr}
F_{x} & =+F_{H} \cos 15^{\circ} & \\
& =+(362.52 \mathrm{~N}) \cos 15^{\circ} & \\
& =+350.17 \mathrm{~N} & \\
F_{y} & =-F \sin 25^{\circ} & \\
& =-(400 \mathrm{~N}) \sin 25^{\circ} & \\
& =-169.047 \mathrm{~N} & F_{y}=-169.0 \mathrm{~N} \triangleleft
\end{array}
$$

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## PROBLEM 2.62

Determine the magnitude and direction of the force $\mathbf{F}=(690 \mathrm{lb}) \mathbf{i}+(300 \mathrm{lb}) \mathbf{j}-(580 \mathrm{lb}) \mathbf{k}$.

## SOLUTION

$$
\begin{array}{rlrl}
\mathbf{F} & =(690 \mathrm{lb}) \mathbf{i}+(300 \mathrm{lb}) \mathbf{j}-(580 \mathrm{lb}) \mathbf{k} & & \\
F & =\sqrt[F]{{ }^{2}+F^{2}{ }_{y}+F_{z}^{2}} & & \\
& =(690 \mathrm{lb})^{2}+(300 \mathrm{lb})^{2}+(-580 \mathrm{lb})^{2} & F & =950 \mathrm{lb} \leftharpoonup \\
& =950 \mathrm{lb} & \theta_{x}=43.4^{\circ} \leftharpoonup \\
\cos \theta_{x} & =\frac{F_{x}}{F}=\frac{690 \mathrm{lb}}{950 \mathrm{lb}} & & \theta_{y}=71.6^{\circ} \leftharpoonup \\
\cos \theta_{y} & =\frac{F_{y}}{F}=\frac{300 \mathrm{lb}}{950 \mathrm{lb}} & \theta_{z}=127.6^{\circ} \leftharpoonup \\
\cos \theta_{z} & =\frac{F_{z}}{F}=\frac{-580 \mathrm{lb}}{950 \mathrm{lb}} &
\end{array}
$$

## PROBLEM 2.63

Determine the magnitude and direction of the force $\mathbf{F}=(650 \mathrm{~N}) \mathbf{i}-(320 \mathrm{~N}) \mathbf{j}+(760 \mathrm{~N}) \mathbf{k}$.

## SOLUTION

$$
\begin{array}{rlrl}
\mathbf{F} & =(650 \mathrm{~N}) \mathbf{i}-(320 \mathrm{~N}) \mathbf{j}+(760 \mathrm{~N}) \mathbf{k} & \\
F & =\sqrt{{ }^{2}{ }_{x}+F^{2}{ }_{y}+F_{z}^{2}} & & \\
& =\sqrt{650 \mathrm{~N})^{2}+(-320 \mathrm{~N})^{2}+(760 \mathrm{~N})^{2}} & F & =1050 \mathrm{~N} \longleftarrow \\
\cos \theta_{x} & =\frac{F_{x}}{F}=\frac{650 \mathrm{~N}}{1050 \mathrm{~N}} & \theta_{x}=51.8^{\circ} \longleftarrow \\
\cos \theta_{y} & =\frac{F_{y}}{F}=\frac{-320 \mathrm{~N}}{1050 \mathrm{~N}} & \theta_{y}=107.7^{\circ} \longleftarrow \\
\cos \theta_{z} & =\frac{F_{z}}{F}=\frac{760 \mathrm{~N}}{1050 \mathrm{~N}} & \theta_{z}=43.6^{\circ} \longleftarrow
\end{array}
$$

## PROBLEM 2.64

A force acts at the origin of a coordinate system in a direction defined by the angles $\theta_{x}=69.3^{\circ}$ and $\theta_{z}=57.9^{\circ}$. Knowing that the $y$ component of the force is -174.0 lb , determine $(a)$ the angle $\theta_{y},(b)$ the other components and the magnitude of the force.

## SOLUTION

$$
\begin{aligned}
\cos ^{2} \theta_{x}+\cos ^{2} \theta_{y}+\cos ^{2} \theta_{z} & =1 \\
\cos ^{2}\left(69.3^{\circ}\right)+\cos ^{2} \theta_{y}+\cos ^{2}\left(57.9^{\circ}\right) & =1 \\
\cos \theta_{y} & = \pm 0.7699
\end{aligned}
$$

(a) Since $F_{y}<0$, we choose $\cos \theta_{y}=-0.7699$ $\therefore \theta_{y}=140.3^{\circ}$
(b)

$$
\begin{array}{cc}
F_{y}=F \cos \theta y & \\
-174.0 \mathrm{lb}=F(-0.7699) & F=226 \mathrm{lb} \\
F=226.0 \mathrm{lb} & F_{x}=79.9 \mathrm{lb} \\
F_{x}=F \cos \theta_{x}=(226.0 \mathrm{lb}) \cos 69.3^{\circ} & F_{z}=120.1 \mathrm{lb}
\end{array}
$$

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## PROBLEM 2.65

A force acts at the origin of a coordinate system in a direction defined by the angles $\theta_{x}=70.9^{\circ}$ and $\theta_{y}=144.9^{\circ}$. Knowing that the $z$ component of the force is -52.0 lb , determine $(a)$ the angle $\theta_{z},(b)$ the other components and the magnitude of the force.

## SOLUTION

$$
\begin{aligned}
\cos ^{2} \theta_{x}+\cos ^{2} \theta_{y}+\cos ^{2} \theta_{z} & =1 \\
\cos ^{2} 70.9+\cos ^{2} 144.9^{\circ}+\cos ^{2} \theta_{z}^{\circ} & =1 \\
\cos \theta_{z} & = \pm 0.47282
\end{aligned}
$$

(a) Since $F_{z}<0$, we choose $\cos \theta_{z}=-0.47282$ $\therefore \theta_{z}=118.2^{\circ}$ 4
(b)

$$
\begin{array}{rlrl}
F_{z} & =F \cos \theta_{z} & \\
-52.0 l b & =F(-0.47282) & \\
F & =110.0 \mathrm{lb} & F=110.0 \mathrm{lb} \triangleleft \\
F_{x} & =F \cos \theta_{x}=(110.0 \mathrm{lb}) \cos 70.9^{\circ} & F_{x}=36.0 \mathrm{lb} \triangleleft \\
F_{y} & =F \cos \theta_{y}=(110.0 \mathrm{lb}) \cos 144.9^{\circ} & F_{y}=-90.0 \mathrm{lb} \triangleleft
\end{array}
$$

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## PROBLEM 2.66

A force acts at the origin of a coordinate system in a direction defined by the angles $\theta_{y}=55^{\circ}$ and $\theta_{z}=45^{\circ}$. Knowing that the $x$ component of the force is -500 lb , determine ( $a$ ) the angle $\theta_{x}$, (b) the other components and the magnitude of the force.

## SOLUTION

(a) We have

$$
\left(\cos \theta_{x}\right)^{2}+\left(\cos \theta_{y}\right)^{2}+\left(\cos \theta_{z}\right)^{2}=1 \Rightarrow\left(\cos \theta_{y}\right)^{2}=1-\left(\cos \theta_{y}\right)^{2}-\left(\cos \theta_{z}\right)^{2}
$$

Since $F_{x}<0$ we must have $\cos \theta_{x}, 0$
Thus, taking the negative square root, from above, we have:

$$
\cos \theta_{x}=-\sqrt[1]{-(\cos 55)^{2}-(\cos 45)^{2}}=0.41353 \quad \theta_{x}=114.4^{\circ} 4
$$

(b) Then:

$$
F=\frac{F_{x}}{\cos \theta_{x}}=\frac{500 \mathrm{lb}}{0.41353}=1209.10 \mathrm{lb} \quad F=1209 \mathrm{lb}
$$

and

$$
\begin{array}{ll}
F_{y}=F \cos \theta_{y}=(1209.10 \mathrm{lb}) \cos 55^{\circ} & F_{y}=694 \mathrm{lb} \\
F_{z}=F \cos \theta_{z}=(1209.10 \mathrm{lb}) \cos 45^{\circ} & F_{z}=855 \mathrm{lb}
\end{array}
$$

## PROBLEM 2.67

A force $\mathbf{F}$ of magnitude 1200 N acts at the origin of a coordinate system. Knowing that $\theta_{x}=65^{\circ}, \theta_{y}=40^{\circ}$, and $F_{z}>0$, determine (a) the components of the force, (b) the angle $\theta_{z}$.

## SOLUTION

$$
\begin{aligned}
\cos ^{2} \theta_{x}+\cos ^{2} \theta_{y}+\cos ^{2} \theta_{z} & =1 \\
\cos ^{2} 65+\cos ^{2} 40^{\circ}+\cos ^{2} \theta_{z}^{\circ} & =1 \\
\cos \theta_{z} & = \pm 0.48432
\end{aligned}
$$

(b) Since $F_{z}>0$, we choose $\cos \theta_{z}=0.48432$, or $\theta_{z}=61.032$
(a)

$$
\begin{array}{ll}
F=1200 \mathrm{~N} & \\
& F_{x}=F \cos \theta_{x}=(1200 \mathrm{~N}) \cos 65^{\circ} \\
& F_{y}=F \cos \theta_{y}=(1200 \mathrm{~N}) \cos 40^{\circ} \\
F_{z} & =F \cos \theta_{z}=(1200 \mathrm{~N}) \cos 61.032^{\circ}
\end{array} F_{y}=919 \mathrm{~N} \longleftarrow 4
$$



## SOLUTION

We have:

$$
\overrightarrow{B A}=+(320 \mathrm{~mm}) \mathbf{i}+(480 \mathrm{~mm}) \mathbf{j}-(360 \mathrm{~mm}) \mathbf{k} \quad B A=680 \mathrm{~mm}
$$

Thus:

$$
\begin{array}{r}
\mathrm{F}_{B}=T_{B A} \lambda_{B A}=T_{B A} \frac{-\overrightarrow{B A}}{B A}=T_{B A}\left(\frac{8}{17}+\frac{12}{17} \mathbf{j}-\frac{9}{17}\right) \\
\left(\frac{8}{17} T_{B A}\right) \mathbf{i}+\left(\frac{12}{17} T_{B A}\right) \mathbf{j}-\left(\frac{9}{17} T_{B A}\right) \mathbf{k}=0
\end{array}
$$

Setting $T_{B A}=408 \mathrm{~N}$ yields,

$$
F_{x}=+192.0 \mathrm{~N}, F_{y}=+288 \mathrm{~N}, F_{z}=-216 \mathrm{~N} \boldsymbol{4}
$$

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## SOLUTION

We have:

$$
\overrightarrow{D A}=-(250 \mathrm{~mm}) \mathbf{i}+(480 \mathrm{~mm}) \mathbf{j}+(360 \mathrm{~mm}) \mathbf{k} \quad D A=650 \mathrm{~mm}
$$

Thus:

$$
\begin{gathered}
\mathrm{F}_{D}=T_{D A} \lambda_{D A}=T_{D A} \frac{-\overrightarrow{D A}}{D A}=T_{D A}\left(-\frac{5}{13} \mathbf{i}+\frac{48}{65} \mathbf{j}+\frac{36}{65} \mathbf{k}\right) \\
-\left(\frac{5}{13} T_{D A}\right) \mathbf{i}+\left(\frac{48}{65} T_{D A}\right) \mathbf{j}+\left(\frac{36}{65} T_{D A}\right) \mathbf{k}=0
\end{gathered}
$$

Setting $T_{D A}=429 \mathrm{~N}$ yields,

$$
F_{x}=-165.0 \mathrm{~N}, F_{y}=+317 \mathrm{~N}, F_{z}=+238 \mathrm{~N} \text { 【 }
$$



## SOLUTION

Cable AB:


$$
A B=74.216 \mathrm{ft} \quad A C=85.590 \mathrm{ft}
$$

$$
\lambda_{A B}=\frac{\overrightarrow{A B}}{A B}=\frac{(-46.765 \mathrm{ft}) \mathbf{i}+(45 \mathrm{ft}) \mathbf{j}+(36 \mathrm{ft}) \mathbf{k}}{74.216 \mathrm{ft}}
$$

$$
\mathbf{T}_{A B}=T_{A B} \lambda_{A B}=\frac{-46.765 \mathbf{i}+45 \mathbf{j}+36 \mathbf{k}}{74.216}
$$

$$
\begin{aligned}
& \left(T_{A B}\right)_{x}=-1.260 \mathrm{kips} \\
& \left(T_{A B}\right)_{y}=+1.213 \mathrm{kips} \\
& \left(T_{A B}\right)_{z}=+0.970 \mathrm{kips}
\end{aligned}
$$



## PROBLEM 2.71

In order to move a wrecked truck, two cables are attached at $A$ and pulled by winches $B$ and $C$ as shown. Knowing that the tension in cable $A C$ is 1.5 kips , determine the components of the force exerted at $A$ by the cable.

## SOLUTION



Cable $A B$ :

$$
\begin{aligned}
& \lambda_{A C}=\frac{\stackrel{-\mathbb{C}}{C}}{A \overline{\bar{C}}} \frac{(-46.765 \mathrm{ft}) \mathbf{i}+(55.8 \mathrm{ft}) \mathbf{j}+(-45 \mathrm{ft}) \mathbf{k}}{85.590 \mathrm{ft}} \\
& \mathbf{T}_{A C}=T_{A C} \lambda_{C}=(1.5 \mathrm{kips}) \frac{-46.765 \mathbf{i}+55.8 \mathbf{j}-45 \mathbf{k}}{85.590}
\end{aligned}
$$

$$
\begin{aligned}
& \left(T_{A C}\right)_{x}=-0.820 \mathrm{kips} \\
& \left(T_{A C}\right)_{y}=+0.978 \mathrm{kips} \\
& \left(T_{A C}\right)_{z}=-0.789 \mathrm{kips}
\end{aligned}
$$



## SOLUTION

$$
\begin{array}{rlrl}
\mathbf{P} & =(300 \mathrm{~N})\left[-\cos 30^{\circ} \sin 15^{\circ} \mathbf{i}+\sin 30^{\circ} \mathbf{j}+\cos 30^{\circ} \cos 15^{\circ} \mathbf{k}\right] & & \\
& =-(67.243 \mathrm{~N}) \mathbf{i}+(150 \mathrm{~N}) \mathbf{j}+(250.95 \mathrm{~N}) \mathbf{k} & & \\
\mathbf{Q} & =(400 \mathrm{~N})\left[\cos 50^{\circ} \cos 20^{\circ} \mathbf{i}+\sin 50^{\circ} \mathbf{j}-\cos 50^{\circ} \sin 20^{\circ} \mathbf{k}\right] & & \\
& =(400 \mathrm{~N})[0.60402 \mathbf{i}+0.76604 \mathbf{j}-0.21985] & & \\
& =(241.61 \mathrm{~N}) \mathbf{i}+(306.42 \mathrm{~N}) \mathbf{j}-(87.939 \mathrm{~N}) \mathbf{k} & \\
\mathbf{R} & =\mathbf{P}+\mathbf{Q} & & \\
& =(174.367 \mathrm{~N}) \mathbf{i}+(456.42 \mathrm{~N}) \mathbf{j}+(163.011 \mathrm{~N}) \mathbf{k} & & \\
R & =(174.367 \mathrm{~N})^{2}+(456.42 \mathrm{~N})^{2}+(163.011 \mathrm{~N})^{2} & \theta_{x}=70.2^{\circ} \measuredangle \\
& =515.07 \mathrm{~N} & & \theta_{y}=27.6^{\circ} \\
\cos \theta_{x} & =\frac{R_{x}=}{R} \frac{174.367 \mathrm{~N}=0.33853}{515.07 \mathrm{~N}} & & \theta_{z}=71.5^{\circ} \\
\cos \theta_{y} & =\frac{R_{y}=456.42 \mathrm{~N}=0.88613}{R} \frac{415.07 \mathrm{~N}}{5} & \\
\cos \theta_{z} & =\frac{R_{z}=}{R} \frac{163.011 \mathrm{~N}=0.31648}{515.07 \mathrm{~N}} &
\end{array}
$$



## PROBLEM 2.73

Find the magnitude and direction of the resultant of the two forces shown knowing that $P=300 \mathrm{~N}$ and $Q=400 \mathrm{~N}$.

## SOLUTION

$$
\begin{aligned}
& \mathbf{P}=(400 \mathrm{~N})\left[-\cos 30^{\circ} \sin 15^{\circ} \mathbf{i}+\sin 30^{\circ} \mathbf{j}+\cos 30^{\circ} \cos 15^{\circ} \mathbf{k}\right] \\
& =-(89.678 \mathrm{~N}) \mathbf{i}+(200 \mathrm{~N}) \mathbf{j}+(334.61 \mathrm{~N}) \mathbf{k} \\
& \mathbf{Q}=(300 \mathrm{~N})\left[\cos 50^{\circ} \cos 20^{\circ} \mathbf{i}+\sin 50^{\circ} \mathbf{j}-\cos 50^{\circ} \sin 20^{\circ} \mathbf{k}\right] \\
& =(181.21 \mathrm{~N}) \mathbf{i}+(229.81 \mathrm{~N}) \mathbf{j}-(65.954 \mathrm{~N}) \mathbf{k} \\
& \mathbf{R}=\mathbf{P}+\mathbf{Q} \\
& =(91.532 \mathrm{~N}) \mathbf{i}+(429.81 \mathrm{~N}) \mathbf{j}+(268.66 \mathrm{~N}) \mathbf{k} \\
& R=(91.532 \mathrm{~N})^{2}+(429.81 \mathrm{~N})^{2}+(268.66 \mathrm{~N})^{2} \\
& =515.07 \mathrm{~N} \\
& \cos \theta_{x}=\frac{R_{x_{x}}=}{R} \frac{91.532 \mathrm{~N}}{515.07 \mathrm{~N}}=0.177708 \\
& \theta_{x}=79.8^{\circ} \text { 《 } \\
& \cos \theta_{y}=\frac{R_{y}}{R}=\frac{429.81 \mathrm{~N}}{515.07 \mathrm{~N}}=0.83447 \\
& \theta_{y}=33.4^{\circ} \\
& \cos \theta_{z}=\frac{R_{z^{\prime}}}{R} \frac{268.66 \mathrm{~N}}{515.07 \mathrm{~N}}=0.52160 \\
& \theta_{z}=58.6^{\circ}
\end{aligned}
$$



## PROBLEM 2.74

Knowing that the tension is 425 lb in cable $A B$ and 510 lb in cable $A C$, determine the magnitude and direction of the resultant of the forces exerted at $A$ by the two cables.

## SOLUTION

$$
\begin{aligned}
& --\bar{A} B=(40 \mathrm{in} .) \mathbf{i}-(45 \mathrm{in} .) \mathbf{j}+(60 \mathrm{in} .) \mathbf{k} \\
& A B=(40 \mathrm{in} .)^{2}+(45 \mathrm{in} .)^{2}+(60 \mathrm{in} .)^{2}=85 \mathrm{in} . \\
& --I C=(100 \mathrm{in} .) \mathbf{i}-(45 \mathrm{in} .) \mathbf{j}+(60 \mathrm{in} .) \mathbf{k} \\
& A C=\left(\sqrt{00 \mathrm{in} .)^{2}+(45 \mathrm{in} .)^{2}+(60 \mathrm{in} .)^{2}}=125 \mathrm{in} .\right.
\end{aligned}
$$

$$
\mathbf{T}_{A B}=T_{A B} \lambda_{A B}=T_{A B} \frac{--\bar{A} B}{A B}=(425 \mathrm{lb})\left[\frac{(40 \mathrm{in} .) \mathbf{i}-(45 \mathrm{in} .) \mathbf{j}+(60 \mathrm{in} .) \mathbf{k}}{85 \mathrm{in} .}\right]
$$

$$
\mathbf{T}_{A B}=(200 \mathrm{lb}) \mathbf{i}-(225 \mathrm{lb}) \mathbf{j}+(300 \mathrm{lb}) \mathbf{k}
$$

$$
\mathbf{T}_{A C}=T_{A C} \lambda_{A C}=T_{A C} \frac{-\overline{A C}}{A C}=(510 \mathrm{lb})\left[\frac{(100 \mathrm{in} .) \mathbf{i}-(45 \mathrm{in} .) \mathbf{j}+(60 \mathrm{in} .) \mathbf{k}}{125 \mathrm{in} .}\right]
$$

$$
\mathbf{T}_{A C}=(408 \mathrm{lb}) \mathbf{i}-(183.6 \mathrm{lb}) \mathbf{j}+(244.8 \mathrm{lb}) \mathbf{k}
$$

$$
\mathbf{R}=\mathbf{T}_{A B}+\mathbf{T}_{A C}=(608) \mathbf{i}-(408.6 \mathrm{lb}) \mathbf{j}+(544.8 \mathrm{lb}) \mathbf{k}
$$

Then:

$$
R=912.92 \mathrm{lb}
$$

$$
R=913 \mathrm{lb}
$$

and

$$
\begin{array}{ll}
\cos \theta_{\bar{x}}=\frac{608 \mathrm{lb}}{912.92 \mathrm{lb}}=0.66599 & \theta_{x}=48.2^{\circ} \\
\cos \theta_{y}=\frac{408.6 \mathrm{lb}}{912.92 \mathrm{lb}}-0.44757 & \theta_{y}=116.6^{\circ} \\
\cos \theta_{z}=\frac{544.8 \mathrm{lb}}{912.92 \mathrm{lb}}=0.59677 & \theta_{z}=53.4^{\circ}
\end{array}
$$

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## PROBLEM 2.75

Knowing that the tension is 510 lb in cable $A B$ and 425 lb in cable $A C$, determine the magnitude and direction of the resultant of the forces exerted at $A$ by the two cables.

## SOLUTION

Then:

$$
R=912.92 \mathrm{lb}
$$

$$
R=913 \mathrm{lb}
$$

and

$$
\begin{array}{ll}
\cos \theta_{\bar{x}}=\frac{580 \mathrm{lb}}{912.92 \mathrm{lb}}=0.63532 & \theta_{x}=50.6^{\circ} . \\
\cos \theta_{y}=\frac{-423 \mathrm{lb}}{912.92 \mathrm{lb}}=-0.46335 & \theta_{y}=117.6^{\circ} . \\
\cos \theta_{z}=\frac{564 \mathrm{lb}}{912.92 \mathrm{lb}}=0.61780 & \theta_{z}=51.8^{\circ} .
\end{array}
$$

$$
\begin{aligned}
& \stackrel{-}{A} B=(40 \mathrm{in} .) \mathbf{i}-(45 \mathrm{in} .) \mathbf{j}+(60 \mathrm{in} .) \mathbf{k} \\
& A B=(40 \mathrm{in} .)^{2}+(45 \mathrm{in} .)^{2}+(60 \mathrm{in} .)^{2}=85 \mathrm{in} . \\
& -H C=(100 \mathrm{in} .) \mathbf{i}-(45 \mathrm{in} .) \mathbf{j}+(60 \mathrm{in} .) \mathbf{k} \\
& A C=\left(\sqrt{00 \mathrm{in} .)^{2}+(45 \mathrm{in} .)^{2}+(60 \mathrm{in} .)^{2}}=125 \mathrm{in} .\right. \\
& \mathbf{T}_{A B}=T_{A B} \lambda_{A B}=T_{A B} \frac{--\mathrm{A} B}{A B}=(510 \mathrm{lb})\left[\frac{(40 \mathrm{in} .) \mathbf{i}-(45 \mathrm{in} .) \mathbf{j}+(60 \mathrm{in} .) \mathbf{k}}{85 \mathrm{in} .}\right] \\
& \mathbf{T}_{A B}=(240 \mathrm{lb}) \mathbf{i}-(270 \mathrm{lb}) \mathbf{j}+(360 \mathrm{lb}) \mathbf{k} \\
& \mathbf{T}_{A C}=T_{A C} \lambda_{A C}=T_{A C} \frac{--[I C}{A C}=(425 \mathrm{lb})\left[\frac{(100 \mathrm{in} .) \mathbf{i}-(45 \mathrm{in} .) \mathbf{j}+(60 \mathrm{in} .) \mathbf{k}}{125 \mathrm{in} .}\right] \\
& \mathbf{T}_{A C}=(340 \mathrm{lb}) \mathbf{i}-(153 \mathrm{lb}) \mathbf{j}+(204 \mathrm{lb}) \mathbf{k} \\
& \mathbf{R}=\mathbf{T}_{A B}+\mathbf{T}_{A C}=(580 \mathrm{lb}) \mathbf{i}-(423 \mathrm{lb}) \mathbf{j}+(564 \mathrm{lb}) \mathbf{k}
\end{aligned}
$$



## SOLUTION

$$
\begin{aligned}
& \overrightarrow{B D}=-(480 \mathrm{~mm}) \mathbf{i}+(510 \mathrm{~mm}) \mathbf{j}-(320 \mathrm{~mm}) \mathbf{k} \\
& B D=(480 \mathrm{~mm})^{2}+(510 \mathrm{~mm})^{2}+(320 \mathrm{~mm})^{2}=770 \mathrm{~mm} \\
& \mathbf{F}_{B D}=T_{B D} \lambda_{B D}=T_{B D} \frac{--\vec{D}}{B D} \\
& =\frac{(385 \mathrm{~N})}{(770 \mathrm{~mm})}[-(480 \mathrm{~mm}) \mathbf{i}+(510 \mathrm{~mm}) \mathbf{j}-(320 \mathrm{~mm}) \mathbf{k}] \\
& =-(240 \mathrm{~N}) \mathbf{i}+(255 \mathrm{~N}) \mathbf{j}-(160 \mathrm{~N}) \mathbf{k} \\
& \overrightarrow{B E}=-(270 \mathrm{~mm}) \mathbf{i}+(400 \mathrm{~mm}) \mathbf{j}-(600 \mathrm{~mm}) \mathbf{k} \\
& B E=(270 \mathrm{~mm})^{2}+(400 \mathrm{~mm})^{2}+(600 \mathrm{~mm})^{2}=770 \mathrm{~mm} \\
& \mathbf{F}_{B E}=T_{B E} \lambda_{B E}=T_{B E} \frac{--\vec{B}}{B E} \\
& =\frac{(385 \mathrm{~N})}{(770 \mathrm{~mm})}[-(270 \mathrm{~mm}) \mathbf{i}+(400 \mathrm{~mm}) \mathbf{j}-(600 \mathrm{~mm}) \mathbf{k}] \\
& =-(135 \mathrm{~N}) \mathbf{i}+(200 \mathrm{~N}) \mathbf{j}-(300 \mathrm{~N}) \mathbf{k} \\
& \mathbf{R}=\mathbf{F}_{B D}+\mathbf{F}_{B E}=-(375 \mathrm{~N}) \mathbf{i}+(455 \mathrm{~N}) \mathbf{j}-(460 \mathrm{~N}) \mathbf{k} \\
& R=\sqrt{(375 \mathrm{~N})^{2}+(455 \mathrm{~N})^{2}+(460 \mathrm{~N})^{2}}=747.83 \mathrm{~N} \quad R=748 \mathrm{~N} \\
& \cos \theta_{x}=\frac{-375 \mathrm{~N}}{747.83 \mathrm{~N}} \quad \theta_{x}=120.1^{\circ} \\
& \cos \theta_{y}=\frac{455 \mathrm{~N}}{747.83 \mathrm{~N}} \quad \theta_{y}=52.5^{\circ} \\
& \cos \theta_{z}=\frac{-460 \mathrm{~N}}{747.83 \mathrm{~N}} \quad \theta_{z}=128.0^{\circ}
\end{aligned}
$$

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## SOLUTION

We have:

$$
\begin{array}{ll}
--\mathbb{I}=-(320 \mathrm{~mm}) \mathbf{i}-(480 \mathrm{~mm}) \mathbf{j}+(360 \mathrm{~mm}) \mathbf{k} & A B=680 \mathrm{~mm} \\
A B=(450 \mathrm{~mm}) \mathbf{i}-(480 \mathrm{~mm}) \mathbf{j}+(360 \mathrm{~mm}) \mathbf{k} & A C=750 \mathrm{~mm} \\
--\mathbb{A C}=(50 \mathrm{~mm}) \mathbf{i}-(480 \mathrm{~mm}) \mathbf{j}-(360 \mathrm{~mm}) \mathbf{k} & A D=650 \mathrm{~mm} \\
A D=(250
\end{array}
$$

Thus:

$$
\begin{aligned}
& \mathbf{T}_{A B}=T_{A B} \lambda_{A B}=T_{A B} \frac{-\bar{A} B}{A B}=\frac{T_{A B}}{680}(-320 \mathbf{i}-480 \mathbf{j}+360 \mathbf{k}) \\
& \mathbf{T}_{A C}=T_{A C} \lambda_{A C}=T_{A C} \frac{A C}{A C}=\frac{54}{750}(450 \mathbf{i}-480 \mathbf{j}+360 \mathbf{k}) \\
& \mathbf{T}_{A D}=T_{A D} \lambda_{A D}=T_{A D} \frac{-\overline{A D}}{A D}=\frac{T_{A D}}{650}(250 \mathbf{i}-480 \mathbf{j}-360 \mathbf{k})
\end{aligned}
$$

Substituting into the Eq. $\mathbf{R}=\Sigma \mathbf{F}$ and factoring $\mathbf{i}, \mathbf{j}, \mathbf{k}$ :

$$
\begin{aligned}
& \mathbf{R}=\left(-\frac{320}{680} T_{A B}+32.40+\frac{250}{650} T_{A D}\right) \mathbf{i} \\
& +\left(\left\lvert\,-\frac{480}{680} T_{A B}-34.560-\frac{480}{650} T_{A D}\right.\right) \mathbf{j} \\
& +\left(\frac{360}{680} T_{A B}+25.920-\frac{360}{650} T_{A D}\right) \mathbf{k}
\end{aligned}
$$

## SOLUTION (Continued)

Since $\mathbf{R}$ is vertical, the coefficients of $\mathbf{i}$ and $\mathbf{k}$ are zero:

$$
\begin{align*}
& \text { i: } \quad-\frac{320}{680} T_{A B}+32.40+\frac{250}{650} T_{A D}=0  \tag{1}\\
& \text { k: } \quad \quad \frac{360}{680} T_{A B}+25.920-\frac{360}{650} T_{A D}=0 \tag{2}
\end{align*}
$$

Multiply (1) by 3.6 and (2) by 2.5 then add:

$$
\begin{aligned}
& -\frac{252}{680} T_{A B}+181.440=0 \\
& T_{A B}=489.60 \mathrm{~N}
\end{aligned}
$$

$$
T_{A B}=490 \mathrm{~N}
$$

Substitute into (2) and solve for $T_{A D}$ :

$$
\begin{aligned}
\frac{360}{680}(489.60 \mathrm{~N})+25.920-\frac{360}{650} T_{A D} & =0 \\
T_{A D} & =514.80 \mathrm{~N}
\end{aligned}
$$

$$
T_{A D}=515 \mathrm{~N}
$$



## SOLUTION



Cable $A B$ :

$$
T_{A B}=183 \mathrm{lb}
$$

$$
\begin{aligned}
& \mathbf{T}_{A B}=T_{A B A B}=T_{A B} \frac{\overrightarrow{A B}}{A B}=(183 \mathrm{lb}) \frac{(-48 \mathrm{in} .) \mathbf{i}+(29 \mathrm{in} .) \mathbf{j}+(24 \mathrm{in} .) \mathbf{k}}{61 \mathrm{in} .} \\
& \mathbf{T}_{A B}=-(144 \mathrm{lb}) \mathbf{i}+(87 \mathrm{lb}) \mathbf{j}+(72 \mathrm{lb}) \mathbf{k}
\end{aligned}
$$

Cable AC:

$$
\begin{aligned}
& \mathbf{T}_{A C}=T_{A C} \lambda_{A C}=T_{A C} \frac{--\overrightarrow{A C}}{A C}=T_{A C} \frac{(-48 \mathrm{in} .) \mathbf{i}+(25 \mathrm{in} .) \mathbf{j}+(-36 \mathrm{in} .) \mathbf{k}}{65 \mathrm{in} .} \\
& \mathbf{T}_{A C}=-\frac{48}{65} T_{A C} \mathbf{i}+\frac{25}{65} T_{A C} \mathbf{j}-\frac{36}{65} T_{A C} \mathbf{k}
\end{aligned}
$$

Load $P$ :

$$
\mathbf{P}=P \mathbf{j}
$$

For resultant to be directed along $O A$, i.e., $x$-axis

$$
R_{z}=0: \Sigma F=(72 \mathrm{lb})-{ }_{z}^{36} \frac{T^{\prime}}{65}=0 \quad T_{A C}=130.0 \mathrm{lb}
$$

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## SOLUTION

See Problem 2.78. Since resultant must be directed along $O A$, i.e., the $x$-axis, we write

$$
R_{y}=0: \Sigma F=(87 \mathrm{lb})+{ }_{y}^{25} \frac{T}{65} \quad A C-P=0
$$

$T_{A C}=130.0 \mathrm{lb}$ from Problem 2.97.

Then

$$
(87 \mathrm{lb})+\frac{25}{65}(130.0 \mathrm{lb})-P=0 \quad P=137.0 \mathrm{lb}
$$



## SOLUTION

## Free-Body Diagram at A:



The forces applied at $A$ are:

$$
\mathbf{T}_{A B}, \mathbf{T}_{A C}, \mathbf{T}_{A D} \text {, and } \mathbf{W}
$$

where $\mathbf{W}=W \mathbf{j}$. To express the other forces in terms of the unit vectors $\mathbf{i}, \mathbf{j}, \mathbf{k}$, we write
and

$$
\begin{aligned}
& \stackrel{H}{A B}=-(450 \mathrm{~mm}) \mathbf{i}+(600 \mathrm{~mm}) \mathbf{j} \\
& A B=750 \mathrm{~mm} \\
& A C=+(600 \mathrm{~mm}) \mathbf{j}-(320 \mathrm{~mm}) \mathbf{k} \\
& -\bar{A} D=+(500 \mathrm{~mm}) \mathbf{i}+(600 \mathrm{~mm}) \mathbf{j}+(360 \mathrm{~mm}) \mathbf{k} \\
& A C=680 \mathrm{~mm} \\
& A D=860 \mathrm{~mm} \\
& \mathbf{T}_{A B}=\lambda_{A B} T_{A B}=T_{A B} \frac{\bar{A} B}{A B}=T_{A B} \frac{(-450 \mathrm{~mm}) \mathbf{i}+(600 \mathrm{~mm}) \mathbf{j}}{750 \mathrm{~mm}} \\
& \left.=\left(-\frac{45}{75} \mathbf{i}+\frac{60}{75} \mathbf{j}\right)\right)_{A B} \\
& \mathbf{T}_{A C}=\lambda_{A C} T_{A C}=T_{A C} \frac{--\bar{A} C}{A C}=T_{A C} \frac{(600 \mathrm{~mm}) \mathbf{i}-(320 \mathrm{~mm}) \mathbf{j}}{680 \mathrm{~mm}} \\
& =\left(\frac{60}{68} \mathbf{j}-\frac{32}{68} \mathbf{k}\right) T_{A C} \\
& \mathbf{T}_{A D}=\lambda_{A D} T_{A D}=T_{A D} \frac{-\bar{A}}{A D}=T_{A D} \frac{(500 \mathrm{~mm}) \mathbf{i}+(600 \mathrm{~mm}) \mathbf{j}+(360 \mathrm{~mm}) \mathbf{k}}{860 \mathrm{~mm}} \\
& =\left(\frac{50}{86} \mathbf{i}+\frac{60}{86} \mathbf{j}+\frac{36}{86} \mathbf{k}\right) T_{A D}
\end{aligned}
$$

## SOLUTION (Continued)

Equilibrium condition:

$$
\Sigma F=0: \therefore \mathbf{T}_{A B}+\mathbf{T}_{A C}+\mathbf{T}_{A D}+\mathbf{W}=0
$$

Substituting the expressions obtained for $\mathbf{T}_{A B}, \mathbf{T}_{A C}$, and $\mathbf{T}_{A D}$; factoring $\mathbf{i}, \mathbf{j}$, and $\mathbf{k}$; and equating each of the coefficients to zero gives the following equations:

From i:

$$
\begin{equation*}
-\frac{45}{75} T_{A B}+\frac{50}{86} T_{A D}=0 \tag{1}
\end{equation*}
$$

From $\mathbf{j}: \quad \frac{60}{75} T_{A B}+\frac{60}{68} T_{A C}+\frac{60}{86} T_{A D}-W=0$

From $\mathbf{k}$ :

$$
\begin{equation*}
-\frac{32}{68} T_{A C}+\frac{36}{86} T_{A D}=0 \tag{3}
\end{equation*}
$$

Setting $T_{A B}=6 \mathrm{kN}$ in (1) and (2), and solving the resulting set of equations gives

$$
\begin{aligned}
& T_{A C}=6.1920 \mathrm{kN} \\
& T_{A C}=5.5080 \mathrm{kN}
\end{aligned}
$$

$$
W=13.98 \mathrm{kN}
$$



## SOLUTION

## Free-Body Diagram at A:



The forces applied at $A$ are:
$\mathbf{T}_{A B}, \mathbf{T}_{A C}, \mathbf{T}_{A D}$, and $\mathbf{W}$
where $\mathbf{W}=W \mathbf{j}$. To express the other forces in terms of the unit vectors $\mathbf{i}, \mathbf{j}, \mathbf{k}$, we write
and

$$
\begin{array}{ll}
--\mathrm{II} B=-(450 \mathrm{~mm}) \mathbf{i}+(600 \mathrm{~mm}) \mathbf{j} & A B=750 \mathrm{~mm} \\
--\mathbb{-}=+(600 \mathrm{~mm}) \mathbf{j}-(320 \mathrm{~mm}) \mathbf{k} & A C=680 \mathrm{~mm} \\
A C=-(500 \mathrm{~mm}) \mathbf{i}+(600 \mathrm{~mm}) \mathbf{j}+(360 \mathrm{~mm}) \mathbf{k} & A D=860 \mathrm{~mm}
\end{array}
$$

$$
\begin{aligned}
\mathbf{T}_{A B}=\lambda_{A B} T_{A B}=T_{A B} \frac{-\bar{A} \mathrm{I}}{A B} & =T_{A B} \frac{(-450 \mathrm{~mm}) \mathbf{i}+(600 \mathrm{~mm}) \mathbf{j}}{750 \mathrm{~mm}} \\
& =\left(-\frac{45}{75} \mathbf{i}+\frac{60}{75} \mathbf{j}\right) f_{A B}
\end{aligned}
$$

$$
\begin{aligned}
\mathbf{T}_{A C}=\lambda_{A C} T_{A C}=T_{A C} \frac{-\bar{A} \mathrm{I} C}{A C} & =T_{A C} \frac{(600 \mathrm{~mm}) \mathbf{i}-(320 \mathrm{~mm}) \mathbf{j}}{680 \mathrm{~mm}} \\
& =\left(\frac{60}{68} \mathbf{j}-\frac{32}{68} \mathbf{k}\right) T_{A C} \\
\mathbf{T}_{A D}=\lambda_{A D} T_{A D}=T_{A D} \frac{-\bar{A} \frac{I}{A D}}{A D} & =T_{A D} \frac{(500 \mathrm{~mm}) \mathbf{i}+(600 \mathrm{~mm}) \mathbf{j}+(360 \mathrm{~mm}) \mathbf{k}}{860 \mathrm{~mm}} \\
& =\left(\frac{50}{86} \mathbf{i}+\frac{60}{86} \mathbf{j}+\frac{36}{86} \mathbf{k}\right) T_{A D}
\end{aligned}
$$

## PROBLEM 2.81 (Continued)

Equilibrium condition:

$$
\Sigma F=0: \therefore \mathbf{T}_{A B}+\mathbf{T}_{A C}+\mathbf{T}_{A D}+\mathbf{W}=0
$$

Substituting the expressions obtained for $\mathbf{T}_{A B}, \mathbf{T}_{A C}$, and $\mathbf{T}_{A D}$; factoring $\mathbf{i}, \mathbf{j}$, and $\mathbf{k}$; and equating each of the coefficients to zero gives the following equations:

Fromi

$$
-\frac{45}{75} T_{A B}+\frac{50}{86} T_{A D}=0
$$

From $\mathbf{j}$ :

$$
\frac{60}{75} T_{A B}+\frac{60}{68} T_{A C}+\frac{60}{86} T_{A D}-W=0
$$

From $\mathbf{k}$ :

$$
-\frac{32}{68} T_{A C}+\frac{36}{86} T_{A D}=0
$$

Setting $T_{A D}=4.3 \mathrm{kN}$ into the above equations gives

$$
\begin{aligned}
& T_{A B}=4.1667 \mathrm{kN} \\
& T_{A C}=3.8250 \mathrm{kN} \quad W=9.71 \mathrm{kN}
\end{aligned}
$$



## SOLUTION



The forces applied at $A$ are:
$\mathbf{T}_{A B}, \mathbf{T}_{A C}, \mathbf{T}_{A D}$, and $\mathbf{P}$
where $\mathbf{P}=P \mathbf{j}$. To express the other forces in terms of the unit vectors $\mathbf{i}, \mathbf{j}, \mathbf{k}$, we write
and

$$
\begin{array}{ll}
--\mathbb{I U}=-(4.20 \mathrm{~m}) \mathbf{i}-(5.60 \mathrm{~m}) \mathbf{j} & A B=7.00 \mathrm{~m} \\
-\bar{A} \mathbb{H}=(2.40 \mathrm{~m}) \mathbf{i}-(5.60 \mathrm{~m}) \mathbf{j}+(4.20 \mathrm{~m}) \mathbf{k} & A C=7.40 \mathrm{~m} \\
-\overline{A D}=-(5.60 \mathrm{~m}) \mathbf{j}-(3.30 \mathrm{~m}) \mathbf{k} & A D=6.50 \mathrm{~m} \\
\mathbf{T}_{A B}=T_{A B} \lambda_{A B}=T_{A B} \frac{-H B}{A B}=(-0.6 \mathbf{i}-0.8 \mathbf{j}) T_{A B} \\
\mathbf{T}_{A C}=T_{A C} \lambda_{A C}=T_{A C} \frac{-\overrightarrow{A C}}{A C}=(0.32432-0.75676 \mathbf{j}+0.56757 \mathbf{k}) T_{A C} \\
\mathbf{T}_{A D}=T_{A D} \lambda_{A D}=T_{A D} \frac{A D}{A D}=(-0.86154 \mathbf{j}-0.50769 \mathbf{k}) T_{A D}
\end{array}
$$

## PROBLEM 2.82 (Continued)

## Equilibrium condition

$$
\Sigma F=0: \mathbf{T}_{A B}+\mathbf{T}_{A C}+\mathbf{T}_{A D}+P \mathbf{j}=0
$$

Substituting the expressions obtained for $\mathbf{T}_{A B}, \mathbf{T}_{A C}$, and $\mathbf{T}_{A D}$ and factoring $\mathbf{i}, \mathbf{j}$, and $\mathbf{k}$ :

$$
\begin{gathered}
\left(-0.6 T_{A B}+0.32432 T_{A C}\right) \mathbf{i}+\left(-0.8 T_{A B}-0.75676 T_{A C}-0.86154 T_{A D}+P\right) \mathbf{j} \\
+\left(0.56757 T_{A C}-0.50769 T_{A D}\right) \mathbf{k}=0
\end{gathered}
$$

Equating to zero the coefficients of $\mathbf{i}, \mathbf{j}, \mathbf{k}$ :

$$
\begin{array}{r}
-0.6 T_{A B}+0.32432 T_{A C}=0 \\
-0.8 T_{A B}-0.75676 T_{A C}-0.86154 T_{A D}+P=0 \\
0.56757 T_{A C}-0.50769 T_{A D}=0 \tag{3}
\end{array}
$$

From Eq. (1)

$$
T_{A B}=0.54053 T_{A C}
$$

From Eq. (3)

$$
T_{A D}=1.11795 T_{A C}
$$

Substituting for $T_{A B}$ and $T_{A D}$ in terms of $T_{A C}$ into Eq. (2) gives:

$$
\begin{aligned}
&-0.8\left(0.54053 T_{A C}\right)-0.75676 T_{A C}-0.86154\left(1.11795 T_{A C}\right)+P=0 \\
& 2.1523 T_{A C}=P ; \quad P=800 \mathrm{~N} \\
& T_{A C}=\frac{800 \mathrm{~N}}{2.1523} \\
&=371.69 \mathrm{~N}
\end{aligned}
$$

Substituting into expressions for $T_{A B}$ and $T_{A D}$ gives:

$$
\begin{aligned}
& T_{A B}=0.54053(371.69 \mathrm{~N}) \\
& T_{A D}=1.11795(371.69 \mathrm{~N})
\end{aligned}
$$

$$
T_{A B}=201 \mathrm{~N}, T_{A C}=372 \mathrm{~N}, T_{A D}=416 \mathrm{~N}
$$

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## SOLUTION

The forces applied at $A$ are:

$$
\mathbf{T}_{A B}, \mathbf{T}_{A C}, \mathbf{T}_{A D} \text { and } \mathbf{W}
$$

where $\mathbf{P}=P \mathbf{j}$. To express the other forces in terms of the unit vectors $\mathbf{i}, \mathbf{j}, \mathbf{k}$, we write
and

$$
\begin{aligned}
& -\bar{A} B=-(36 \mathrm{in} .) \mathbf{i}+(60 \mathrm{in} .) \mathbf{j}-(27 \mathrm{in} .) \mathbf{k} \\
& A B=75 \mathrm{in} . \\
& -H C=(60 \mathrm{in} .) \mathbf{j}+(32 \mathrm{in} .) \mathbf{k} \\
& A C=68 \mathrm{in} . \\
& -H D=(40 \mathrm{in} .) \mathbf{i}+(60 \mathrm{in} .) \mathbf{j}-(27 \mathrm{in} .) \mathbf{k} \\
& A D=77 \mathrm{in} .
\end{aligned}
$$

$$
\begin{aligned}
\mathbf{T}_{A B} & =T_{A B} \lambda_{A B}=T_{A B} \frac{-\bar{A} B}{A B} \\
& =(-0.48 \mathbf{i}+0.8 \mathbf{j}-0.36 \mathbf{k}) T_{A B} \\
\mathbf{T}_{A C} & =T_{A C} \lambda_{A C}=T_{A C} \frac{--\bar{A} C}{A C} \\
& =(0.88235 \mathbf{j}+0.47059 \mathbf{k}) T_{A C} \\
\mathbf{T}_{A D} & =T_{A D} \lambda_{A D}=T_{A D} \frac{--\bar{A} D}{A D} \\
& =(0.51948 \mathbf{i}+0.77922 \mathbf{j}-0.35065 \mathbf{k}) T_{A D}
\end{aligned}
$$

Equilibrium Condition with $\quad \mathbf{W}=-W \mathbf{j}$

$$
\Sigma F=0: \mathbf{T}_{A B}+\mathbf{T}_{A C}+\mathbf{T}_{A D}-W \mathbf{j}=0
$$

## PROBLEM 2.83 (Continued)

Substituting the expressions obtained for $\mathbf{T}_{A B}, \mathbf{T}_{A C}$, and $\mathbf{T}_{A D}$ and factoring $\mathbf{i}, \mathbf{j}$, and $\mathbf{k}$ :

$$
\begin{gathered}
\left(-0.48 T_{A B}+0.51948 T_{A D}\right) \mathbf{i}+\left(0.8 T_{A B}+0.88235 T_{A C}+0.77922 T_{A D}-W\right) \mathbf{j} \\
+\left(-0.36 T_{A B}+0.47059 T_{A C}-0.35065 T_{A D}\right) \mathbf{k}=0 \\
-0.48 T_{A B}+0.51948 T_{A D}=0 \\
0.8 T_{A B}+0.88235 T_{A C}+0.77922 T_{A D}-W=0 \\
-0.36 T_{A B}+0.47059 T_{A C}-0.35065 T_{A D}=0
\end{gathered}
$$

Substituting $T_{A D}=616 \mathrm{lb}$ in Equations (1), (2), and (3) above, and solving the resulting set of equations ${ }_{A B}$ usig $87 . g n Y$ Yntional algorithms, gives:

$$
T_{A C}=969.00 \mathrm{lb} W=1868 \mathrm{lb}
$$



## SOLUTION

See Problem 2.83 for the figure and the analysis leading to the linear algebraic Equations (1), (2), and (3) below:

$$
\begin{array}{r}
-0.48 T_{A B}+0.51948 T_{A D}=0 \\
0.8 T_{A B}+0.88235 T_{A C}+0.77922 T_{A D}-W=0 \\
-0.36 T_{A B}+0.47059 T_{A C}-0.35065 T_{A D}=0 \tag{3}
\end{array}
$$

Substituting $T_{A C}=544 \mathrm{lb}$ in Equations (1), (2), and (3) above, and solving the resulting set of equations using conventional algorithms, gives:

$$
\begin{aligned}
& T_{A B}=374.27 \mathrm{lb} \\
& T_{A D}=345.82 \mathrm{lb}
\end{aligned} \quad W=1049 \mathrm{lb}
$$

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## SOLUTION

See Problem 2.83 for the figure and the analysis leading to the linear algebraic Equations (1), (2), and (3) below:

$$
\begin{array}{r}
-0.48 T_{A B}+0.51948 T_{A D}=0 \\
0.8 T_{A B}+0.88235 T_{A C}+0.77922 T_{A D}-W=0 \\
-0.36 T_{A B}+0.47059 T_{A C}-0.35065 T_{A D}=0 \tag{3}
\end{array}
$$

Substituting $W=1600 \mathrm{lb}$ in Equations (1), (2), and (3) above, and solving the resulting set of equations using conventional algorithms, gives

$$
\begin{aligned}
T_{A B} & =571 \mathrm{lb} \\
T_{A C} & =830 \mathrm{lb} \\
T_{A D} & =528 \mathrm{lb}
\end{aligned}
$$



## PROBLEM 2.86

Three wires are connected at point $D$, which is located 18 in. below the T-shaped pipe support $A B C$. Determine the tension in each wire when a $180-\mathrm{lb}$ cylinder is suspended from point $D$ as shown.

## SOLUTION

Free-Body Diagram of Point $D$ :


The forces applied at $D$ are:

$$
\mathbf{T}_{D A}, \mathbf{T}_{D B}, \mathbf{T}_{D C} \text { and } \mathbf{W}
$$

where $\mathbf{W}=-180.0 \mathrm{lbj}$. To express the other forces in terms of the unit vectors $\mathbf{i}, \mathbf{j}$, $\mathbf{k}$, we write

$$
\begin{aligned}
& -\bar{D} A=(18 \mathrm{in} .) \mathbf{j}+(22 \mathrm{in} .) \mathbf{k} \\
& D A=28.425 \mathrm{in} . \\
& -\mathbb{B}=-(24 \mathrm{in} .) \mathbf{i}+(18 \mathrm{in} .) \mathbf{j}-(16 \mathrm{in} .) \mathbf{k} \\
& D B=34.0 \mathrm{in} . \\
& ---\mathbb{D} C=(24 \mathrm{in} .) \mathbf{i}+(18 \mathrm{in} .) \mathbf{j}-(16 \mathrm{in} .) \mathbf{k} \\
& D C=34.0 \mathrm{in} .
\end{aligned}
$$

## SOLUTION (Continued)

$$
\begin{aligned}
\mathbf{T}_{D A} & =T_{D a} \lambda_{D A}=T_{D a} \frac{---\mathbb{D}}{D A} \\
& =(0.63324 \mathbf{j}+0.77397 \mathbf{k}) T_{D A} \\
\mathbf{T}_{D B} & =T_{D B} \lambda_{D B}=T_{D B} \frac{-U B}{D B} \\
& =(-0.70588 \mathbf{i}+0.52941 \mathbf{j}-0.47059 \mathbf{k}) T_{D B} \\
\mathbf{T}_{D C} & =T_{D C} \lambda_{D C}=T_{D C} \frac{D C}{D C} \\
& =(0.70588 \mathbf{i}+0.52941 \mathbf{j}-0.47059 \mathbf{k}) T_{D C}
\end{aligned}
$$

and

Equilibrium Condition with

$$
\mathbf{W}=-W \mathbf{j}
$$

$$
\Sigma F=0: \mathbf{T}_{D A}+\mathbf{T}_{D B}+\mathbf{T}_{D C}-W \mathbf{j}=0
$$

Substituting the expressions obtained for $\mathbf{T}_{D A}, \mathbf{T}_{D B}$, and $\mathbf{T}_{D C}$ and factoring $\mathbf{i}, \mathbf{j}$, and $\mathbf{k}$ :

$$
\begin{array}{r}
\left(-0.70588 T_{D B}+0.70588 T_{D C}\right) \mathbf{i} \\
\left(0.63324 T_{D A}+0.52941 T_{D B}+0.52941 T_{D C}-W\right) \mathbf{j} \\
\left(0.77397 T_{D A}-0.47059 T_{D B}-0.47059 T_{D C}\right) \mathbf{k}
\end{array}
$$

Equating to zero the coefficients of $\mathbf{i}, \mathbf{j}, \mathbf{k}$ :

$$
\begin{array}{r}
-0.70588 T_{D B}+0.70588 T_{D C}=0 \\
0.63324 T_{D A}+0.52941 T_{D B}+0.52941 T_{D C}-W=0 \\
0.77397 T_{D A}-0.47059 T_{D B}-0.47059 T_{D C}=0 \tag{3}
\end{array}
$$

Substituting $W=180 \mathrm{lb}$ in Equations (1), (2), and (3) above, and solving the resulting set of equations using conventional algorithms gives,

$$
\begin{aligned}
T_{D A} & =119.7 \mathrm{lb} \\
T_{D B} & =98.4 \mathrm{lb} \\
T_{D C} & =98.4 \mathrm{lb}
\end{aligned}
$$

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## SOLUTION

By Symmetry $T_{D B}=T_{D C}$

## Free-Body Diagram of Point $D$ :



The forces applied at $D$ are:
$\mathbf{T}_{D B}, \mathbf{T}_{D C}, \mathbf{T}_{D A}$, and $\mathbf{P}$
where $\mathbf{P}=P \mathbf{j}=(36 \mathrm{lb}) \mathbf{j}$. To express the other forces in terms of the unit vectors $\mathbf{i}, \mathbf{j}, \mathbf{k}$, we write
and

$$
\begin{aligned}
& --\bar{D} A=(16 \mathrm{in} .) \mathbf{i}-(24 \mathrm{in} .) \mathbf{j} \quad D A=28.844 \mathrm{in} . \\
& D B=-(8 \mathrm{in} .) \mathbf{i}-(24 \mathrm{in} .) \mathbf{j}+(6 \mathrm{in} .) \mathbf{k} \quad D B=26.0 \mathrm{in} . \\
& \stackrel{-U}{D C}=-(8 \mathrm{in}) \mathbf{i}-(24 \mathrm{in} .) \mathbf{j}-(6 \mathrm{in} .) \mathbf{k} \quad D C=26.0 \mathrm{in} . \\
& \mathbf{T}_{D A}=T_{D A} \lambda_{D A}=T_{D A} \frac{--\| A}{D A}=(0.55471 \mathbf{i}-0.83206 \mathbf{j}) T_{D A} \\
& \mathbf{T}_{D B}=T_{D B} \lambda_{D B}=T_{D B} \frac{-\overline{D B}}{D B}=(-0.30769 \mathbf{i}-0.92308 \mathbf{j}+0.23077 \mathbf{k}) T_{D B} \\
& \mathbf{T}_{D C}=T_{D C} \lambda_{D C}=T_{D C} \frac{\overline{D C}}{D C}=(-0.30769 \mathbf{i}-0.92308 \mathbf{j}-0.23077 \mathbf{k}) T_{D C}
\end{aligned}
$$

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## SOLUTION (Continued)

## Equilibrium condition:

$$
\Sigma F=0: \mathbf{T}_{D A}+\mathbf{T}_{D B}+\mathbf{T}_{D C}+(36 \mathrm{lb}) \mathbf{j}=0
$$

Substituting the expressions obtained for $\mathbf{T}_{D A}, \mathbf{T}_{D B}$, and $\mathbf{T}_{D C}$ and factoring $\mathbf{i}, \mathbf{j}$, and $\mathbf{k}$ :

$$
\begin{aligned}
\left(0.55471 T_{D A}-0.30769 T_{D B}-\right. & \left.0.30769 T_{D C}\right) \mathbf{i}+\left(-0.83206 T_{D A}-0.92308 T_{D B}-0.92308 T_{D C}+36 \mathrm{lb}\right) \mathbf{j} \\
& +\left(0.23077 T_{D B}-0.23077 T_{D C}\right) \mathbf{k}=0
\end{aligned}
$$

Equating to zero the coefficients of $\mathbf{i}, \mathbf{j}, \mathbf{k}$ :

$$
\begin{array}{r}
0.55471 T_{D A}-0.30769 T_{D B}-0.30769 T_{D C}=0 \\
-0.83206 T_{D A}-0.92308 T_{D B}-0.92308 T_{D C}+36 \mathrm{lb}=0 \\
0.23077 T_{D B}-0.23077 T_{D C}=0 \tag{3}
\end{array}
$$

Equation (3) confirms that $T_{D B}=T_{D C}$. Solving simultaneously gives,

$$
T_{D A}=14.42 \mathrm{lb} ; \quad T_{D B}=T_{D C}=13.00 \mathrm{lb}
$$

## PROBLEM 2.88

A rectangular plate is supported by three cables as shown. Knowing that the tension in cable $A C$ is 60 N , determine the weight of the plate.

## SOLUTION

We note that the weight of the plate is equal in magnitude to the force $\mathbf{P}$ exerted by the support on Point $A$.

Free Body $\boldsymbol{A}$ :

$$
\Sigma F=0: \mathbf{T}_{A B}+\mathbf{T}_{A C}+\mathbf{T}_{A D}+P \mathbf{j}=0
$$

We have:

$$
\begin{array}{ll}
---\amalg & \\
\underline{A} B_{\perp}=-(320 \mathrm{~mm}) \mathbf{i}-(480 \mathrm{~mm}) \mathbf{j}+(360 \mathrm{~mm}) \mathbf{k} & A B=680 \mathrm{~mm} \\
\underset{A}{A} C_{\perp}=(450 \mathrm{~mm}) \mathbf{i}-(480 \mathrm{~mm}) \mathbf{j}+(360 \mathrm{~mm}) \mathbf{k} & A C=750 \mathrm{~mm} \\
A D=(250 \mathrm{~mm}) \mathbf{i}-(480 \mathrm{~mm}) \mathbf{j}-(360 \mathrm{~mm}) \mathbf{k} & A D=650 \mathrm{~mm}
\end{array}
$$

Thus:


$$
\begin{aligned}
& \left.\mathbf{T}_{A B}=T_{A B} \lambda_{A B}=T_{A B} \frac{--\bar{A} B}{A B}=\left(-\frac{8}{17}-\frac{12}{17} \mathbf{j}+\frac{9}{17}\right)_{T}\right)_{A B} \\
& \mathbf{T}_{A C}=T_{A C} \lambda_{A C}=T_{A C} \frac{-H C}{A C}=(0.6 \mathbf{i}-0.64 \mathbf{j}+0.48 \mathbf{k}) T_{A C} \\
& \mathbf{T}_{A D}=T_{A D} \lambda_{A D}=T_{A D} \frac{--I D}{A D}=\left(\frac{5}{13}-\frac{9.6}{13} \mathbf{j}-\frac{7.2}{13} \mathbf{k}\right) T_{A D}
\end{aligned}
$$

Substituting into the Eq. $\Sigma F=0$ and factoring $\mathbf{i}, \mathbf{j}, \mathbf{k}$ :

$$
\begin{aligned}
& \left(-\frac{8}{17} T_{A B}+0.6 T_{A C}+\frac{5}{13} T{ }_{A D}\right) \mathbf{i} \\
& +\left(-\frac{12}{17} T_{A B}-0.64 T_{A C}-\frac{9.6}{13} T_{A D}+P\right) \mathbf{j} \\
& +\left(\frac{9}{17} T_{A B}+0.48 T_{A C}-\frac{7.2}{13} T_{A D}\right) \mathbf{k}=0
\end{aligned}
$$

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## SOLUTION (Continued)

Setting the coefficient of $\mathbf{i}, \mathbf{j}, \mathbf{k}$ equal to zero:

$$
\begin{align*}
& \text { i: } \quad-\frac{8}{17} T_{A B}+0.6 T_{A C}+\frac{5}{13} T_{A D}=0  \tag{1}\\
& \text { j: } \quad \tag{2}
\end{align*} \quad-\frac{12}{7} T_{A B}-0.64 T_{A C}-\frac{9.6}{13} T_{A D}+P=0 .
$$

Making $T_{A C}=60 \mathrm{~N}$ in (1) and (3):

$$
\begin{align*}
& -\frac{8}{17} T_{A B}+36 \mathrm{~N}+\frac{5}{13} T_{A D}=0  \tag{1'}\\
& \frac{9}{17} T_{A B}+28.8 \mathrm{~N}-\frac{7.2}{13} T_{A D}=0 \tag{3'}
\end{align*}
$$

Multiply ( $1^{\prime}$ ) by $9,\left(3^{\prime}\right)$ by 8 , and add:

$$
554.4 \mathrm{~N}-\frac{12.6}{13} T_{A D}=0 \quad T_{A D}=572.0 \mathrm{~N}
$$

Substitute into ( $1^{\prime}$ ) and solve for $T_{A B}$ :

$$
T_{A B}=\frac{17}{8}\left(36+\frac{5}{13} \times 572\right) \quad T_{A B}=544.0 \mathrm{~N}
$$

Substitute for the tensions in Eq. (2) and solve for $P$ :

$$
P=\frac{12}{17}(544 \mathrm{~N})+0.64(60 \mathrm{~N})+\frac{9.6}{13}(572 \mathrm{~N})
$$

$$
=844.8 \mathrm{~N} \quad \text { Weight of plate }=P=845 \mathrm{~N}
$$



## SOLUTION

See Problem 2.88 for the figure and the analysis leading to the linear algebraic Equations (1), (2), and (3) below:

$$
\begin{align*}
-\frac{8}{17} T_{A B}+0.6 T_{A C}+\frac{5}{13} T_{A D} & =0  \tag{1}\\
-\frac{12}{17} T_{A B}+0.64 T_{A C}-\frac{9.6}{13} T_{A D}+P & =0  \tag{2}\\
\frac{9}{17} T_{A B}+0.48 T_{A C}-\frac{7.2}{13} T_{A D} & =0 \tag{3}
\end{align*}
$$

Making $T_{A D}=520 \mathrm{~N}$ in Eqs. (1) and (3):

$$
\begin{align*}
&-\frac{8}{17} T_{A B}+0.6 T_{A C}+200 \mathrm{~N}=0  \tag{1'}\\
& \frac{9}{17} T_{A B}+0.48 T_{A C}-288 \mathrm{~N}=0 \tag{3'}
\end{align*}
$$

Multiply (1') by $9,\left(3^{\prime}\right)$ by 8 , and add:

$$
9.24 T_{A C}-504 \mathrm{~N}=0 T_{A C}=54.5455 \mathrm{~N}
$$

Substitute into ( $1^{\prime}$ ) and solve for $T_{A B}$ :

$$
T_{A B}=\frac{17}{8}(0.6 \times 54.5455+200) \quad T_{A B}=494.545 \mathrm{~N}
$$

Substitute for the tensions in Eq. (2) and solve for $P$ :

$$
\begin{aligned}
P & =\frac{12}{17}(494.545 \mathrm{~N})+0.64(54.5455 \mathrm{~N})+\frac{9.6}{13}(520 \mathrm{~N}) \\
& =768.00 \mathrm{~N}
\end{aligned}
$$

$$
\text { Weight of plate }=P=768 \mathrm{~N}
$$

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## PROBLEM 2.90

In trying to move across a slippery icy surface, a $175-\mathrm{lb}$ man uses two ropes $A B$ and $A C$. Knowing that the force exerted on the man by the icy surface is perpendicular to that surface, determine the tension in each rope.

## SOLUTION

## Free-Body Diagram at $A$

$$
I_{A B}
$$

$$
\begin{aligned}
\mathbf{T}_{A C}=T_{A C} \lambda_{A C}=T_{A C} \frac{--\operatorname{AC}}{A C} & =T_{A C} \frac{(-30 \mathrm{ft}) \mathbf{i}+(20 \mathrm{ft}) \mathbf{j}-(12 \mathrm{ft}) \mathbf{k}}{38 \mathrm{ft}} \\
& =T_{A C}\left(-\frac{15}{19} \mathbf{i}+\frac{10}{19} \mathbf{j}-\frac{6}{19} \mathbf{k}\right) \\
\mathbf{T}_{A B}=T_{A B} \lambda_{A B}=T_{A B} \frac{\bar{A} B}{A B} & =T_{A B} \frac{(-30 \mathrm{ft}) \mathbf{i}+(24 \mathrm{ft}) \mathbf{j}+(32 \mathrm{ft}) \mathbf{k}}{50 \mathrm{ft}} \\
& =T_{A B}\left(-\frac{15}{25} \mathbf{i}+\frac{12}{25} \mathbf{j}+\frac{16}{25} \mathbf{k}\right)
\end{aligned}
$$

Equilibrium condition: $\Sigma \mathbf{F}=0$

$$
\mathbf{T}_{A B}+\mathbf{T}_{A C}+\mathbf{N}+\mathbf{W}=0
$$

## SOLUTION (Continued)

Substituting the expressions obtained for $\mathbf{T}_{A B}, \mathbf{T}_{A C}, \mathbf{N}$, and $\mathbf{W}$; factoring $\mathbf{i}, \mathbf{j}$, and $\mathbf{k}$; and equating each of the coefficients to zero gives the following equations:

From i:

$$
\begin{equation*}
-\frac{15}{25} T_{A B}-\frac{15}{19} T_{A C}+\frac{16}{34} N=0 \tag{1}
\end{equation*}
$$

From $\mathbf{j}: \quad \frac{12}{25} T_{A B}+\frac{10}{19} T_{A C}+\frac{30}{34} N-(175 \mathrm{lb})=0$

From k:

$$
\begin{equation*}
\frac{16}{25} T_{A B}-\frac{6}{19}_{A C}=0 \tag{2}
\end{equation*}
$$

Solving the resulting set of equations gives:

$$
T_{A B}=30.8 \mathrm{lb} ; T_{A C}=62.5 \mathrm{lb}
$$



## SOLUTION

Refer to Problem 2.90 for the figure and analysis leading to the following set of equations, Equation (3) being modified to include the additional force $\mathbf{P}=(-45 \mathrm{lb}) \mathbf{k}$.

$$
\begin{align*}
-\frac{15}{25} T_{A B}-\frac{15}{19} T_{A C}+\frac{16}{34} N & =0  \tag{1}\\
\frac{12}{25} T_{A B}+\frac{10}{19} T_{A C}+\frac{30}{34} N-(175 \mathrm{lb}) & =0  \tag{2}\\
\frac{16}{25} T_{A B}-\frac{6}{19} T_{A C}-(45 \mathrm{lb}) & =0 \tag{3}
\end{align*}
$$

Solving the resulting set of equations simultaneously gives:

$$
\begin{aligned}
T_{A B} & =81.3 \mathrm{lb} \\
T_{A C} & =22.2 \mathrm{lb}
\end{aligned}
$$

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## SOLUTION

$$
\begin{aligned}
& \Sigma \mathbf{F}_{A}=0: \quad \mathbf{T}_{A B}+\mathbf{T}_{A C}+\mathbf{T}_{A D}+\mathbf{P}=0 \quad \text { where } \quad \mathbf{P}=P \mathbf{i} \\
& -\stackrel{-1}{A B}=-(960 \mathrm{~mm}) \mathbf{i}-(240 \mathrm{~mm}) \mathbf{j}+(380 \mathrm{~mm}) \mathbf{k} \quad A B=1060 \mathrm{~mm} \\
& \bar{A} C=-(960 \mathrm{~mm}) \mathbf{i}-(240 \mathrm{~mm}) \mathbf{j}-(320 \mathrm{~mm}) \mathbf{k} \quad A C=1040 \mathrm{~mm} \\
& -\vec{A} D=-(960 \mathrm{~mm}) \mathbf{i}+(720 \mathrm{~mm}) \mathbf{j}-(220 \mathrm{~mm}) \mathbf{k} \quad A D=1220 \mathrm{~mm} \\
& \mathbf{T}_{A B}=T_{A B} \lambda_{A B}=T_{A B} \frac{-\bar{A} B}{A B}=T_{A B}\left(-\frac{48}{53} \mathbf{i}-\frac{12}{53} \mathbf{j}+\frac{19}{53} \mathbf{k}\right) \\
& \mathbf{T}_{A C}=T_{A C} \lambda_{A C}=T_{A C} \frac{--\bar{A} C}{A C}=T_{A C}\left(-\frac{12}{13} \mathbf{i} \frac{3}{13} \mathbf{j}-\frac{4}{13} \mathbf{k}\right) \\
& \mathbf{T}_{A D}=T_{A D} \text { ÂD }=\frac{305 \mathrm{~N}}{1220 \mathrm{~mm}}[(-960 \mathrm{~mm}) \mathbf{i}+(720 \mathrm{~mm}) \mathbf{j}-(220 \mathrm{~mm}) \mathbf{k}] \\
& =-(240 \mathrm{~N}) \mathbf{i}+(180 \mathrm{~N}) \mathbf{j}-(55 \mathrm{~N}) \mathbf{k}
\end{aligned}
$$

Substituting into $\Sigma \mathbf{F}_{A}=0$, factoring $\mathbf{i}, \mathbf{j}, \mathbf{k}$, and setting each coefficient equal to $\phi$ gives:

$$
\begin{align*}
& \mathbf{i}: P=\frac{48}{53}{ }_{A B}+\frac{12}{13} T_{A C}+240 \mathrm{~N}  \tag{1}\\
& \mathbf{j}: \quad \frac{12}{53} T_{A B}+\frac{3}{13} T_{A C}=180 \mathrm{~N}  \tag{2}\\
& \mathbf{k}: \quad \frac{19}{53} T_{A B}-\frac{4}{13} T_{A C}=55 \mathrm{~N} \tag{3}
\end{align*}
$$

Solving the system of linear equations using conventional algorithms gives:

$$
\begin{array}{ll}
T_{A B}=446.71 \mathrm{~N} & P=960 \mathrm{~N} \\
T_{A C}=341.71 \mathrm{~N}
\end{array}
$$

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## SOLUTION

We assume that $T_{A D}=0$ and write $\quad \Sigma \mathbf{F}_{A}=0: \mathbf{T}_{A B}+\mathbf{T}_{A C}+Q \mathbf{j}+(1200 \mathrm{~N}) \mathbf{i}=0$

$$
\begin{aligned}
& A B=-(960 \mathrm{~mm}) \mathbf{i}-(240 \mathrm{~mm}) \mathbf{j}+(380 \mathrm{~mm}) \mathbf{k} \quad A B=1060 \mathrm{~mm} \\
& A C=-(960 \mathrm{~mm}) \mathbf{i}-(240 \mathrm{~mm}) \mathbf{j}-(320 \mathrm{~mm}) \mathbf{k} A C=1040 \mathrm{~mm} \\
& \mathbf{T}_{A B}=T_{A B} \lambda_{A B}=T_{A B} \frac{--\mathbb{A}}{A B}=\left(-\frac{48}{53} \mathbf{i}-\frac{12}{53} \mathbf{j}+\frac{19}{53} \mathbf{k}\right) T_{A B} \\
& \mathbf{T}_{A C}=T_{A C} \lambda_{A C}=T_{A C} \frac{--\overline{A C}}{A C}=\left(-\frac{12}{13} \mathbf{i}-\frac{3}{13} \mathbf{j}-\frac{4}{13} \mathbf{k}\right) T_{A C}
\end{aligned}
$$

Substituting into $\Sigma \mathbf{F}_{A}=0$, factoring $\mathbf{i}, \mathbf{j}, \mathbf{k}$, and setting each coefficient equal to $\phi$ gives:

$$
\begin{align*}
& \mathbf{i}: \quad-\frac{48}{53} T_{A B}-\frac{12}{13} T_{A C}+1200 \mathrm{~N}=0  \tag{1}\\
& \mathbf{j}: \quad-\frac{12}{53} T_{A B}-\frac{3}{13} T_{A C}+Q=0  \tag{2}\\
& \mathbf{k}: \quad \frac{19}{53} T_{A B}-\frac{4}{13} T_{A C}=0 \tag{3}
\end{align*}
$$

Solving the resulting system of linear equations using conventional algorithms gives:

$$
\begin{aligned}
T_{A B} & =605.71 \mathrm{~N} \\
T_{A C} & =705.71 \mathrm{~N} \\
Q & =300.00 \mathrm{~N}
\end{aligned}
$$

$$
0 \leq Q<300 \mathrm{~N} \leftharpoonup
$$

Note: This solution assumes that $Q$ is directed upward as shown $(Q \geq 0)$, if negative values of $Q$ are considered, cable $A D$ remains taut, but $A C$ becomes slack for $Q=-460 \mathrm{~N}$.

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## SOLUTION

$$
\begin{aligned}
\mathbf{T}_{A B} & =T \lambda_{A B} \\
& =T \frac{\overline{A B}}{A B} \\
& =T \frac{(-130 \mathrm{~mm}) \mathbf{i}+(400 \mathrm{~mm}) \mathbf{j}+(160 \mathrm{~mm}) \mathbf{k}}{450 \mathrm{~mm}} \\
& =T\left(-\frac{13}{45} \mathbf{i}+\frac{40}{45} \mathbf{j}+\frac{16}{45} \mathbf{k}\right) \\
\mathbf{T}_{A C} & =T \lambda_{A C} \\
& =T \frac{\bar{C}}{A C} \\
& =T \underline{(-150 \mathrm{~mm}) \mathbf{i}+(400 \mathrm{~mm}) \mathbf{j}+(-240 \mathrm{~mm}) \mathbf{k}} \\
& =T\left(-\frac{15}{49} \mathbf{i}+\frac{40}{49} \mathbf{j}-\frac{24}{49} \mathbf{k}\right) \\
\Sigma & =0: \mathbf{T}_{A B}+\mathbf{T}_{A C}+\mathbf{Q}+\mathbf{P}+\mathbf{W}=0
\end{aligned}
$$

$$
\text { Free-Body } A \text { : }
$$

$$
\underline{I}_{A B}=T \quad T_{A C}=\underline{T}
$$

Setting coefficients of $\mathbf{i}, \mathbf{j}, \mathbf{k}$ equal to zero:

$$
\begin{array}{ll}
\mathbf{i}:-\frac{13}{45} T-\frac{15}{49} T+P=0 & 0.59501 T=P \\
\mathbf{j}:+\frac{40}{45}+\frac{40}{49}-W=0 & 1.70521 T=W \\
\text { k : }+\frac{16}{45} T-\frac{24}{49} T+Q=0 & 0.134240 T=Q \tag{3}
\end{array}
$$

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## PROBLEM 2.94 (Continued)

Data:

$$
\begin{array}{rlr}
W=376 \mathrm{~N} & 1.70521 T=376 \mathrm{~N} & T=220.50 \mathrm{~N} \\
0.59501(220.50 \mathrm{~N})=P & & P=131.2 \mathrm{~N} \triangleleft \\
0.134240(220.50 \mathrm{~N})=Q & & Q=29.6 \mathrm{~N}
\end{array}
$$



## SOLUTION

Refer to Problem 2.94 for the figure and analysis resulting in Equations (1), (2), and (3) for $P, W$, and $Q$ in terms of $T$ below. Setting $P=164 \mathrm{~N}$ we have:
Eq. (1):
$0.59501 T=164 \mathrm{~N}$

$$
T=275.63 \mathrm{~N}
$$

Eq. (2):
$1.70521(275.63 \mathrm{~N})=W$
$W=470 \mathrm{~N}$ 4
Eq. (3):
$0.134240(275.63 \mathrm{~N})=Q$
$Q=37.0 \mathrm{~N} 4$

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## PROBLEM 2.96

Cable $B A C$ passes through a frictionless ring $A$ and is attached to fixed supports at $B$ and $C$, while cables $A D$ and $A E$ are both tied to the ring and are attached, respectively, to supports at $D$ and $E$. Knowing that a $200-\mathrm{lb}$ vertical $\operatorname{load} \mathbf{P}$ is applied to ring $A$, determine the tension in each of the three cables.

## SOLUTION

## Free Body Diagram at $\boldsymbol{A}$ :

Since $T_{B A C}=$ tension in cable $B A C$, it follows that

$$
T_{A B}
$$

$$
\begin{gathered}
T_{A B}=T_{A C}=T_{B A C} \\
\left.\mathbf{T}_{A B}=T_{B A C} \lambda_{A B}=T_{B A C} \frac{(-17.5 \mathrm{in} .) \mathbf{i}+(60 \mathrm{in} .) \mathbf{j}}{62.5 \mathrm{in} .}=T_{B A C}\left(\frac{-17.5}{62.5} \mathbf{i}+\frac{60}{62.5}\right)^{\prime}\right) \\
\mathbf{T}_{A C}=T_{B A C} \lambda_{A C}=T_{B A C} \frac{(60 \mathrm{in} .) \mathbf{i}+(25 \mathrm{in} .) \mathbf{k}}{65 \mathrm{in} .}=T_{B A C}\left(\frac{60}{65} \mathbf{j}+\frac{25}{65} \mathbf{k}\right) \\
\mathbf{T}_{A D}=T_{A D} \lambda_{A D}=T_{A D} \frac{(80 \mathrm{in} .) \mathbf{i}+(60 \mathrm{in} .) \mathbf{i}}{100 \mathrm{in} .}=T_{A D}\left(\frac{4}{5} \mathbf{i}+\left.\frac{3}{5} \mathbf{j}\right|_{j}\right. \\
\mathbf{T}_{A E}=T_{A E} \lambda_{A E}=T_{A E} \frac{(60 \mathrm{in} .) \mathbf{j}-(45 \mathrm{in} .) \mathbf{k}}{75 \mathrm{in} .}=T_{A E}\left(\frac{4}{5} \mathbf{j}-\frac{3}{5} \mathbf{k}\right)
\end{gathered}
$$

## SOLUTION Continued

Substituting into $\Sigma \mathbf{F}_{A}=0$, setting $\mathbf{P}=(-200 \mathrm{lb}) \mathbf{j}$, and setting the coefficients of $\mathbf{i}, \mathbf{j}, \mathbf{k}$ equal to $\phi$, we obtain the following three equilibrium equations:

From

$$
\begin{equation*}
\text { i: } \quad-\frac{17.5}{62.5} T_{B A C}+\frac{4}{5} T_{A D}=0 \tag{1}
\end{equation*}
$$

From $\quad \mathbf{j}:\left(\frac{60+}{62.5} \frac{60)_{T}}{65}\right){ }_{B A C}+\frac{3}{5} T_{A D}+\frac{4}{5} T_{A E}-200 \mathrm{lb}=0$
From

$$
\begin{equation*}
\mathbf{k}: \frac{25}{65} T_{B A C}-\frac{3}{5} T_{A E}=0 \tag{2}
\end{equation*}
$$

Solving the system of linear equations using conventional algorithms gives:

$$
T_{B A C}=76.7 \mathrm{lb} ; T_{A D}=26.9 \mathrm{lb} ; T_{A E}=49.2 \mathrm{lb}
$$



## PROBLEM 2.97

Knowing that the tension in cable $A E$ of Prob. 2.96 is 75 lb , determine ( $a$ ) the magnitude of the load $\mathbf{P},(b)$ the tension in cables $B A C$ and $A D$.

PROBLEM 2.96 Cable $B A C$ passes through a frictionless ring $A$ and is attached to fixed supports at $B$ and $C$, while cables $A D$ and $A E$ are both tied to the ring and are attached, respectively, to supports at $D$ and $E$. Knowing that a $200-\mathrm{lb}$ vertical load $\mathbf{P}$ is applied to ring $A$, determine the tension in each of the three cables.

## SOLUTION

Refer to the solution to Problem 2.96 for the figure and analysis leading to the following set of equilibrium equations, Equation (2) being modified to include $P \mathbf{j}$ as an unknown quantity:

$$
\begin{align*}
& -\frac{17.5}{62.5} T_{B A C}+\frac{4}{5} T_{A D}=0 \\
& \left(\frac{60}{62.5}+\frac{60}{65}\right)_{B A C}+\frac{3}{5} T_{A D}+\frac{4}{5} T_{A E}-P=0 \\
& \frac{25}{65} T_{B A C}-\frac{3}{5} T_{A E}=0 \tag{3}
\end{align*}
$$

Substituting for $T_{A E}=75 \mathrm{lb}$ and solving simultaneously gives:
(a)
$P=305 \mathrm{lb}$
(b) $\quad T_{B A C}=117.0 \mathrm{lb} ; T_{A D}=40.9 \mathrm{lb}$

## SOLUTION Continued

Then from the specifications of the problem, $y=155 \mathrm{~mm}=0.155 \mathrm{~m}$

$$
\begin{aligned}
z^{2} & =0.23563 \mathrm{~m}^{2}-(0.155 \mathrm{~m})^{2} \\
z & =0.46 \mathrm{~m}
\end{aligned}
$$

and
(a)

$$
\begin{aligned}
T_{A B} & =\frac{341 \mathrm{~N}}{0.155(1.90476)} \\
& =1155.00 \mathrm{~N}
\end{aligned}
$$

or

$$
T_{A B}=1155 \mathrm{~N}
$$

and
(b)

$$
\begin{aligned}
Q & =\frac{341 \mathrm{~N}(0.46 \mathrm{~m})(0.866)}{(0.155 \mathrm{~m})} \\
& =(1012.00 \mathrm{~N})
\end{aligned}
$$

or


## PROBLEM 2.98

A container of weight $W$ is suspended from ring $A$, to which cables $A C$ and $A E$ are attached. A force $\mathbf{P}$ is applied to the end $F$ of a third cable that passes over a pulley at $B$ and through ring $A$ and that is attached to a support at $D$. Knowing that $W=1000 \mathrm{~N}$, determine the magnitude of $P$. (Hint: The tension is the same in all portions of cable $F B A D$.)

## SOLUTION

The (vector) force in each cable can be written as the product of the (scalar) force and the unit vector along the cable. That is, with
and
and

$$
\begin{aligned}
\overrightarrow{A B} & =-(0.78 \mathrm{~m}) \mathbf{i}+(1.6 \mathrm{~m}) \mathbf{j}+(0 \mathrm{~m}) \mathbf{k} \\
A B & =\sqrt{(-0.78 \mathrm{~m})^{2}+(1.6 \mathrm{~m})^{2}+(0)^{2}} \\
& =1.78 \mathrm{~m} \\
\mathbf{T}_{A B} & =T \lambda_{A B}=T_{A B} \frac{--\overrightarrow{A B}}{A B} \\
& =\frac{T_{A B}}{1.78 \mathrm{~m}}[-(0.78 \mathrm{~m}) \mathbf{i}+(1.6 \mathrm{~m}) \mathbf{j}+(0 \mathrm{~m}) \mathbf{k}] \\
\mathbf{T}_{A B} & =T_{A B}(-0.4382 \mathbf{i}+0.8989 \mathbf{j}+0 \mathbf{k}) \\
\overrightarrow{A C} & (0) \mathbf{i}+(1.6 \mathrm{~m}) \mathbf{j}+(1.2 \mathrm{~m}) \mathbf{k} \\
A C & =(\sqrt{\mathrm{m}})^{2}+(1.6 \mathrm{~m})^{2}+(1.2 \mathrm{~m})^{2}=2 \mathrm{~m} \\
\mathbf{T}_{A C} & =T \lambda_{A C}=T_{A C} \frac{---\overrightarrow{A C}}{A C}=\frac{T_{A C}}{2 \mathrm{~m}}[(0) \mathbf{i}+(1.6 \mathrm{~m}) \mathbf{j}+(1.2 \mathrm{~m}) \mathbf{k}] \\
\mathbf{T}_{A C} & =T_{A C}(0.8 \mathbf{j}+0.6 \mathbf{k}) \\
\overrightarrow{A D} & =(1.3 \mathrm{~m}) \mathbf{i}+(1.6 \mathrm{~m}) \mathbf{j}+(0.4 \mathrm{~m}) \mathbf{k} \\
A D & =\left(\sqrt{1.3 \mathrm{~m})^{2}+(1.6 \mathrm{~m})^{2}+(0.4 \mathrm{~m})^{2}=} 2.1 \mathrm{~m}\right. \\
\mathbf{T}_{A D} & =T \lambda_{A D}=T_{A D} \frac{--\overrightarrow{A D}}{A D}=\frac{T}{2 D}[(1.3 \mathrm{~m}) \mathbf{i}+(1.6 \mathrm{~m}) \mathbf{j}+(0.4 \mathrm{~m}) \mathbf{k}] \\
\mathbf{T}_{A D} & =T_{A D}(0.6190 \mathbf{i}+0.7619 \mathbf{j}+0.1905 \mathbf{k})
\end{aligned}
$$

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## SOLUTION Continued

Finally,

$$
\begin{aligned}
\overrightarrow{A E} & -(0.4 \mathrm{~m}) \mathbf{i}+(1.6 \mathrm{~m}) \mathbf{j}-(0.86 \mathrm{~m}) \mathbf{k} \\
A E & =\sqrt{(-0.4 \mathrm{~m})^{2}+(1.6 \mathrm{~m})^{2}+(-0.86 \mathrm{~m})^{2}}=1.86 \mathrm{~m} \\
\mathbf{T}_{A E} & =T \lambda_{A E}=T_{A E} \frac{--\overrightarrow{A E}}{A E} \\
& =\frac{T_{A E}}{1.86 \mathrm{~m}}[-(0.4 \mathrm{~m}) \mathbf{i}+(1.6 \mathrm{~m}) \mathbf{j}-(0.86 \mathrm{~m}) \mathbf{k}] \\
\mathbf{T}_{A E} & =T_{A E}(-0.2151 \mathbf{i}+0.8602 \mathbf{j}-0.4624 \mathbf{k})
\end{aligned}
$$

With the weight of the container

$$
\begin{aligned}
& \mathbf{W}=-W \mathbf{j}, \text { at } A \text { wehave: } \\
& \Sigma \mathbf{F}=0: \mathbf{T}_{A B}+\mathbf{T}_{A C}+\mathbf{T}_{A D}-W \mathbf{j}=0
\end{aligned}
$$

Equating the factors of $\mathbf{i}, \mathbf{j}$, and $\mathbf{k}$ to zero, we obtain the following linear algebraic equations:

$$
\begin{array}{r}
-0.4382 T_{A B}+0.6190 T_{A D}-0.2151 T_{A E}=0 \\
0.8989 T_{A B}+0.8 T_{A C}+0.7619 T_{A D}+0.8602 T_{A E}-W=0 \\
0.6 T_{A C}+0.1905 T_{A D}-0.4624 T_{A E}=0 \tag{3}
\end{array}
$$

Knowing that $W=1000 \mathrm{~N}$ and that because of the pulley system at $B T_{A B}=T_{A D}=P, \quad$ where $P$ is the externally applied (unknown) force, we can solve the system of linear Equations (1), (2) and (3) uniquely for $P$.

$$
P=378 \mathrm{~N}
$$



## SOLUTION

From the geometry of the chute:

$$
\begin{aligned}
\mathbf{N} & \left.=\frac{N}{\sqrt{5}} 2 \mathbf{j}+\mathbf{k}\right) \\
& =N(0.8944 \mathbf{j}+0.4472 \mathbf{k})
\end{aligned}
$$

The force in each rope can be written as the product of the magnitude of the force and the unit vector along the cable. Thus, with

$$
\begin{aligned}
\overrightarrow{A B} & =(40 \mathrm{in} .) \mathbf{i}+(70 \mathrm{in} .) \mathbf{j}-(40 \mathrm{in} .) \mathbf{k} \\
A B & =\sqrt{40 \mathrm{in} .)^{2}+(70 \mathrm{in} .)^{2}+(40 \mathrm{in} .)^{2}} \\
& =90 \mathrm{in.} \\
\mathbf{T}_{A B} & =T \lambda_{A B}=T_{A B} \frac{-\overrightarrow{A B}}{A B} \\
& =\frac{T_{A B}}{90}[(-40 \mathrm{in} .) \mathbf{i}+(70 \mathrm{in} .) \mathbf{j}-(40 \mathrm{in} .) \mathbf{k}] \\
\mathbf{T}_{A B} & =T_{A B}\left(-\frac{4}{9} \mathbf{i}+\frac{7}{9} \mathbf{j}-\frac{4}{9} \mathbf{k}\right)
\end{aligned}
$$

and

$$
\overrightarrow{A C}=(45 \mathrm{in} .) \mathbf{i}+(60 \mathrm{in} .) \mathbf{j}-(40 \mathrm{in} .) \mathbf{k}
$$

$$
A C=(45 \mathrm{in} .)^{2}+(60 \mathrm{in} .)^{2}+(40 \mathrm{in} .)^{2}=85 \mathrm{in} .
$$

$$
\mathbf{T}_{A C}=T \lambda_{A C}=T_{A C} \frac{\overrightarrow{A C}}{A C}
$$

$$
=\frac{T_{A C}}{85 \mathrm{in} .}[(45 \mathrm{in} .) \mathbf{i}+(60 \mathrm{in} .) \mathbf{j}-(40 \mathrm{in} .) \mathbf{k}]
$$

$$
\mathbf{T}_{A C}=T_{A C}\left(\frac{9}{17}+\frac{12}{17} \mathbf{j}-\frac{8}{17}\right)
$$

Then:

$$
\Sigma \mathbf{F}=0: \mathbf{N}+\mathbf{T}_{A B}+\mathbf{T}_{A C}+\mathbf{W}=0
$$

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## SOLUTION Continued

With $W=200 \mathrm{lb}$, and equating the factors of $\mathbf{i}, \mathbf{j}$, and $\mathbf{k}$ to zero, we obtain the linear algebraic equations:

$$
\begin{align*}
& \text { i: } \quad-\frac{4}{9} T_{A B}+\frac{9}{17} T{ }_{A C}=0  \tag{1}\\
& \text { j: } \quad \frac{7}{9} T_{A B}+\frac{12}{17} T_{A C}+\frac{2}{\sqrt{5}} 200 \mathrm{lb}=0  \tag{2}\\
& \text { k: } \quad-\frac{4}{9} T{ }_{A B}-\frac{8}{17} T{ }_{A C}+\frac{1}{\sqrt{5}}=0 \tag{3}
\end{align*}
$$

Using conventional methods for solving linear algebraic equations we obtain:

$$
\begin{gathered}
T_{A B}=65.6 \mathrm{lb} \\
T_{A C}=55.1 \mathrm{lb}
\end{gathered}
$$



## SOLUTION

## Free-Body Diagrams of Collars:

A:


B:


$$
\lambda_{A B}=\frac{\overrightarrow{A B}}{A B}=\frac{-x \mathbf{i}-(20 \mathrm{in} .) \mathbf{j}+z \mathbf{k}}{25 \mathrm{in} .}
$$

Collar A:

$$
\Sigma \mathbf{F}=0: P \mathbf{i}+N_{y} \mathbf{j}+N_{z} \mathbf{k}+T_{A B} \lambda_{A B}=0
$$

Substitute for $\lambda_{A B}$ and set coefficient of $\mathbf{i}$ equal to zero:

$$
\begin{equation*}
P-\frac{T_{A B} x}{25 \mathrm{in} .}=0 \tag{1}
\end{equation*}
$$

Collar B:

$$
\Sigma \mathbf{F}=0: \quad(60 \mathrm{lb}) \mathbf{k}+N_{x}^{\prime} \mathbf{i}+N_{y}^{\prime} \mathbf{j}-T_{A B} \lambda_{A B}=0
$$

Substitute for $\lambda_{A B}$ and set coefficient of $\mathbf{k}$ equal to zero:

$$
\begin{equation*}
60 \mathrm{lb}-\frac{T_{A B} z}{25 \mathrm{in} .}=0 \tag{2}
\end{equation*}
$$

(a)

$$
\begin{array}{r}
x=9 \text { in. } \quad(9 \mathrm{in} .)^{2}+(20 \mathrm{in} .)^{2}+z^{2}=(25 \mathrm{in} .)^{2} \\
z=12 \mathrm{in} .
\end{array}
$$

From Eq. (2):
$\frac{60 \mathrm{lb}-T_{A B}}{25} \frac{(12 \mathrm{in} .)}{}$

$$
P=\frac{(125.0 \mathrm{lb})(9 \mathrm{in} .)}{25 \mathrm{in} .}
$$

$$
P=45.0 \mathrm{lb} \measuredangle
$$

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## SOLUTION

See Problem 2.100 for the diagrams and analysis leading to Equations (1) and (2) below:

$$
\begin{array}{r}
P=\frac{T_{A B} x}{25 \mathrm{in.}}=0 \\
60 \mathrm{lb}-\frac{T_{A B} z}{25 \mathrm{in.}}=0 \tag{2}
\end{array}
$$

For $P=120 \mathrm{lb}$, Eq. (1) yields

$$
\begin{align*}
T_{A B} x & =(25 \mathrm{in} .)(20 \mathrm{lb})  \tag{1'}\\
T_{A B} z & =(25 \mathrm{in} .)(60 \mathrm{lb})  \tag{2'}\\
\frac{x}{z} & =2
\end{align*}
$$

From Eq. (2):

Dividing Eq. (1') by (2'),

$$
\begin{equation*}
x^{2}+z^{2}+(20 \mathrm{in} .)^{2}=(25 \mathrm{in} .)^{2} \tag{4}
\end{equation*}
$$

Solving (3) and (4) simultaneously,

From Eq. (3):

$$
\begin{aligned}
4 z^{2}+z^{2}+400 & =625 \\
z^{2} & =45 \\
z & =6.7082 \mathrm{in}
\end{aligned}
$$

$$
\begin{aligned}
x & =2 z=2(6.7082 \mathrm{in} .) \\
& =13.4164 \mathrm{in} .
\end{aligned}
$$

$$
x=13.42 \mathrm{in} ., \quad z=6.71 \mathrm{in}
$$

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## PROBLEM 2.102

Collars $A$ and $B$ are connected by a $525-\mathrm{mm}$-long wire and can slide freely on frictionless rods. If a force $\mathbf{P}=(341 \mathrm{~N}) \mathbf{j}$ is applied to collar $A$, determine $(a)$ the tension in the wire when $y=155 \mathrm{~mm}$, $(b)$ the magnitude of the force $\mathbf{Q}$ required to maintain the equilibrium of the system.

## SOLUTION

For both Problems 2.102 and 2.103:
Free-Body Diagrams of Collars:

$$
(A B)^{2}=x^{2}+y^{2}+z^{2}
$$

Here

$$
(0.525 \mathrm{~m})^{2}=(0.20 \mathrm{~m})^{2}+y^{2}+z^{2}
$$

or

$$
y^{2}+z^{2}=0.23563 \mathrm{~m}^{2}
$$

Thus, when $y$ given, $z$ is determined,
Now

$$
\begin{aligned}
\lambda_{A B} & =\frac{\overrightarrow{A B}}{A B} \\
& =\frac{1}{0.525 \mathrm{~m}}(0.20 \mathbf{i}-y \mathbf{j}+z \mathbf{k}) \mathrm{m} \\
& =0.38095 \mathbf{i}-1.90476 \mathbf{y} \mathbf{j}+1.90476 z \mathbf{k}
\end{aligned}
$$




Where $y$ and $z$ are in units of meters, $m$.
From the F.B. Diagram of collar $A: \quad \Sigma \mathbf{F}=0: N_{x} \mathbf{i}+N_{z} \mathbf{k}+P \mathbf{j}+T_{A B} \lambda_{A B}=0$
Setting the $\mathbf{j}$ coefficient to zero gives $\quad P-(1.90476 y) T_{A B}=0$
With

$$
P=341 \mathrm{~N}
$$

$$
T_{A B}=\frac{341 \mathrm{~N}}{1.90476 y}
$$

Now, from the free body diagram of collar $B$ :

$$
\Sigma \mathbf{F}=0: N_{x} \mathbf{i}+N_{y} \mathbf{j}+Q \mathbf{k}-T_{A B} \lambda_{A B}=0
$$

Setting the $\mathbf{k}$ coefficient to zero gives $Q-T_{A B}(1.90476 z)=0$

And using the above result for $T_{A B}$ we have

$$
Q=T_{A \bar{B}}=\frac{341 \mathrm{~N}}{(1.90476) y}(1.90476 z)=\frac{(341 \mathrm{~N})(z)}{y}
$$

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## PROBLEM 2.103

Solve Problem 2.102 assuming that $y=275 \mathrm{~mm}$.

PROBLEM 2.102 Collars $A$ and $B$ are connected by a $525-\mathrm{mm}$-long wire and can slide freely on frictionless rods. If a force $\mathbf{P}=(341 \mathrm{~N}) \mathbf{j}$ is applied to collar $A$, determine (a) the tension in the wire when $y=155 \mathrm{~mm}$, (b) the magnitude of the force $\mathbf{Q}$ required to maintain the equilibrium of the system.

## SOLUTION

From the analysis of Problem 2.102, particularly the results:

$$
\begin{aligned}
y^{2}+z^{2} & =0.23563 \mathrm{~m}^{2} \\
T_{A B} & =\frac{341 \mathrm{~N}}{1.90476 y} \\
Q & =\frac{341 \mathrm{~N}}{y} z
\end{aligned}
$$

With $y=275 \mathrm{~mm}=0.275 \mathrm{~m}$, we obtain:

$$
\begin{aligned}
z^{2} & =0.23563 \mathrm{~m}^{2}-(0.275 \mathrm{~m})^{2} \\
z & =0.40 \mathrm{~m}
\end{aligned}
$$

and
(a)

$$
T_{A B}=\frac{341 \mathrm{~N}}{(1.90476)(0.275 \mathrm{~m})}=651.00
$$

or

$$
T_{A B}=651 \mathrm{~N}
$$

and
(b)

$$
Q=\frac{341 \mathrm{~N}(0.40 \mathrm{~m})}{(0.275 \mathrm{~m})}
$$

or $Q=496 \mathrm{~N}$

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## PROBLEM 2.104

Two structural members $A$ and $B$ are bolted to a bracket as shown. Knowing that both members are in compression and that the force is 15 kN in member $A$ and 10 kN in member $B$, determine by trigonometry the magnitude and direction of the resultant of the forces applied to the bracket by members $A$ and $B$.

## SOLUTION

Using the force triangle and the laws of cosines and sines,
we have

$$
\begin{aligned}
& \gamma=180^{\circ}-\left(40^{\circ}+20^{\circ}\right) \\
& =120^{\circ}
\end{aligned}
$$

Then

$$
\begin{aligned}
R^{2}= & (15 \mathrm{kN})^{2}+(10 \mathrm{kN})^{2} \\
& -2(15 \mathrm{kN})(10 \mathrm{kN}) \cos 120^{\circ} \\
= & 475 \mathrm{kN}^{2} \\
R= & 21.794 \mathrm{kN}
\end{aligned}
$$


and

$$
\frac{10 \mathrm{kN}}{\sin \alpha}=\frac{21.794 \mathrm{kN}}{\sin 120^{\circ}}
$$

$$
\sin \alpha=\left(\frac{10 \mathrm{kN}}{21.794 \mathrm{kN}}\right) \sin 120^{\circ}
$$

$$
=0.39737
$$

$$
\alpha=23.414
$$

Hence:
$\phi=\alpha+50^{\circ}=73.414$ $\mathbf{R}=21.8 \mathrm{kN} \times 73.4^{\circ}$


## SOLUTION

Compute the following distances:

$$
\begin{aligned}
O A & =(24 \mathrm{in} .)^{2}+(45 \mathrm{in} .)^{2} \\
& =51.0 \mathrm{in} . \\
O B & =(28 \mathrm{in} .)^{2}+(45 \mathrm{in} .)^{2} \\
& =53.0 \mathrm{in} . \\
O C & =(40 \mathrm{in} .)^{2}+(30 \mathrm{in} .)^{2} \\
& =50.0 \mathrm{in} .
\end{aligned}
$$

102-lb Force:

$$
\begin{aligned}
& F_{x}=-102 \mathrm{lb} \frac{24 \mathrm{in} .}{51.0 \mathrm{in} .} \\
& F_{y}=+102 \mathrm{lb} \frac{45 \mathrm{in} .}{51.0 \mathrm{in} .}
\end{aligned}
$$



$$
F_{x}=-48.0 \mathrm{lb}
$$

$$
F_{y}=+90.0 \mathrm{lb}
$$

106-lb Force

$$
F_{x}=+106 \mathrm{lb} \frac{28 \mathrm{in} .}{53.0 \mathrm{in} .}
$$

$$
F_{x}=+56.0 \mathrm{lb}
$$

$$
F_{y}=+106 \mathrm{lb} \frac{45 \mathrm{in} .}{53.0 \mathrm{in} .}
$$

$$
F_{y}=+90.0 \mathrm{lb}
$$

$$
F_{x}=-200 \mathrm{lb} \frac{40 \mathrm{in} .}{50.0 \mathrm{in} .}
$$

$$
F_{x}=-160.0 \mathrm{lb}
$$

$$
F_{y}=-200 \mathrm{lb} \frac{30 \mathrm{in} .}{50.0 \mathrm{in}}
$$

$$
F_{y}=-120.0 \mathrm{lb}
$$

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## SOLUTION

(a)

$$
\begin{aligned}
180^{\circ} & =45^{\circ}+\alpha+90^{\circ}+30^{\circ} \\
\alpha & =180^{\circ}-45^{\circ}-90^{\circ}-30^{\circ} \\
& =15^{\circ} \\
\cos \alpha & =\frac{P_{x}}{P} \\
P & =\frac{P_{x}}{\cos \alpha} \\
& =\frac{600 \mathrm{~N}}{\cos 15^{\circ}} \\
& =621.17 \mathrm{~N} \quad A
\end{aligned}
$$

(b)

$$
\begin{aligned}
\tan \alpha & =\frac{P_{v}}{P_{x}} \\
P_{y} & =P_{x} \tan \alpha \\
& =(600 \mathrm{~N}) \tan 15^{\circ} \\
& =160.770 \mathrm{~N}
\end{aligned}
$$

$$
P_{y}=160.8 \mathrm{~N}
$$



## SOLUTION

60-lb Force:

$$
\begin{aligned}
& F_{x}=(60 \mathrm{lb}) \cos 20^{\circ}=56.382 \mathrm{lb} \\
& F_{y}=(60 \mathrm{lb}) \sin 20^{\circ}=
\end{aligned}
$$

80-lb Force: $\quad F_{x}=(80 \mathrm{lb}) \cos 60^{\circ}=40.000 \mathrm{lb}$ $F_{y}=(80 \mathrm{lb}) \sin 60^{\circ}=$

120-lb Force:

$$
\begin{aligned}
& F_{x}=(120 \mathrm{lb}) \cos 30^{\circ}= \\
& F_{y}=-(120 \mathrm{lb}) \sin 30^{\circ}=-60.000 \mathrm{lb}
\end{aligned}
$$


and
$R_{x}=\Sigma F_{x}=200.305 \mathrm{lb}$
$R_{y}=\Sigma F_{y}=29.803 \mathrm{lb}$

$$
\begin{aligned}
R & =\left(\sqrt{200.305 \mathrm{lb})^{2}+(29.803 \mathrm{lb})^{2}}\right. \\
& =202.510 \mathrm{lb}
\end{aligned}
$$

Further: $\quad \tan \alpha=\frac{29.803}{200.305}$

$$
\alpha=\tan ^{-1} \frac{29.803}{200.305}
$$

$$
=8.46^{\circ} \quad \mathbf{R}=203 \mathrm{lb} \subset 8.46^{\circ}
$$

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## SOLUTION

Free-Body Diagram


## Force Triangle



Law of sines:

$$
\frac{T_{A C}}{\sin 110^{\circ}}=\frac{T_{B C}}{\sin 5^{\circ}}=\frac{1200 \mathrm{lb}}{\sin 65^{\circ}}
$$

(a)

$$
\begin{array}{ll}
T_{A C}=\frac{1200 \mathrm{lb}}{\sin 65^{\circ}} \sin 110^{\circ} & T_{A C}=1244 \mathrm{lb} \\
T_{B C}=\frac{1200 \mathrm{lb}}{\sin 65^{\circ}} \sin 5^{\circ} & T_{B C}=115.4 \mathrm{lb}
\end{array}
$$

(b)

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## SOLUTION

## Free-Body Diagram



$$
T_{A C}=T_{B C}=800 \mathrm{~N}
$$

$$
\begin{gathered}
A C=B C=\left(\sqrt{\left.h^{2}+d^{2}\right)}\right. \\
\Sigma F_{y}=0: \quad 2(800 \mathrm{~N}) \frac{h}{\sqrt{h^{2}+d^{2}}}-P=0
\end{gathered}
$$

$$
800=\frac{P}{2} \sqrt{1+\left(\frac{d}{h}\right)^{2}}
$$

Data: $\quad P=200 \mathrm{~N}, d=600 \mathrm{~mm}$ and solving for $h$

$$
800 \mathrm{~N}=\frac{200 \mathrm{~N}}{2} \sqrt{1+\left(\frac{600 \mathrm{~mm}}{h}\right)^{2}}
$$

$$
h=75.6 \mathrm{~mm} \text { < }
$$

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## PROBLEM 2.110

Three forces are applied to a bracket as shown. The directions of the two 150-N forces may vary, but the angle between these forces is always $50^{\circ}$. Determine the range of values of $\alpha$ for which the magnitude of the resultant of the forces acting at $A$ is less than 600 N .

## SOLUTION

Combine the two $150-\mathrm{N}$ forces into a resultant force $Q$ :


Equivalent loading at $A$ :


Using the law of cosines:

$$
\begin{aligned}
& \quad(600 \mathrm{~N})^{2}=(500 \mathrm{~N})^{2}+(271.89 \mathrm{~N})^{2}+2(500 \mathrm{~N})(271.89 \mathrm{~N}) \cos \left(55^{\circ}+\alpha\right) \\
& \cos \left(55^{\circ}+\alpha\right)=0.132685
\end{aligned}
$$

Two values for $\alpha: \quad 55^{\circ}+\alpha=82.375$

$$
\alpha=27.4^{\circ}
$$

or

$$
\begin{aligned}
55^{\circ}+\alpha & =-82.375^{\circ} \\
55^{\circ}+\alpha & =360^{\circ}-82.375^{\circ} \\
\alpha & =222.6^{\circ}
\end{aligned}
$$

For $R<600 \mathrm{lb}$ :
$27.4^{\circ}<\alpha<222.6$

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## SOLUTION



From triangle $A O B: \quad \quad \cos \theta_{y}=\frac{56 \mathrm{ft}}{65 \mathrm{ft}}$

$$
=0.86154
$$

$$
\theta_{y}=30.51^{\circ}
$$

(a)

$$
\begin{aligned}
F_{x} & =-F \sin \theta_{y} \cos 20^{\circ} \\
& =-(3900 \mathrm{lb}) \sin 30.51^{\circ} \cos 20^{\circ}
\end{aligned}
$$

$$
F_{x}=-1861 \mathrm{lb}
$$

$$
F_{y}=+F \cos \theta_{y}=(3900 \mathrm{lb})(0.86154)
$$

$$
F_{y}=+3360 \mathrm{lb}
$$

$$
F_{z}=+(3900 \mathrm{lb}) \sin 30.51^{\circ} \sin 20^{\circ}
$$

$$
F_{z}=+677 \mathrm{lb}
$$

(b)

$$
\begin{array}{rlr}
\cos \theta_{x}=\frac{F_{x}}{F}=-\frac{1861 \mathrm{lb}}{3900 \mathrm{lb}}=-0.4771 & \theta_{x}=118.5^{\circ} \\
\text { From above: } & \theta_{y}=30.51^{\circ} & \theta_{y}=30.5^{\circ} . \\
& \cos \theta_{z}=\frac{F_{z}}{F}=+\frac{677 \mathrm{lb}}{3900 \mathrm{lb}}=+0.1736 & \theta_{z}=80.0^{\circ}
\end{array}
$$

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## SOLUTION



The forces applied at $A$ are:

$$
\mathbf{T}_{A B}, \mathbf{T}_{A C}, \mathbf{T}_{A D} \text {, and } \mathbf{P}
$$

where $\mathbf{P}=P \mathbf{j}$. To express the other forces in terms of the unit vectors $\mathbf{i}, \mathbf{j}, \mathbf{k}$, we write
and

$$
\begin{array}{ll}
-\bar{A} B=-(4.20 \mathrm{~m}) \mathbf{i}-(5.60 \mathrm{~m}) \mathbf{j} & A B=7.00 \mathrm{~m} \\
A C=(2.40 \mathrm{~m}) \mathbf{i}-(5.60 \mathrm{~m}) \mathbf{j}+(4.20 \mathrm{~m}) \mathbf{k} & A C=7.40 \mathrm{~m} \\
-\mathbb{H}=-(5.60 \mathrm{~m}) \mathbf{j}-(3.30 \mathrm{~m}) \mathbf{k} & A D=6.50 \mathrm{~m} \\
\mathbf{T}_{A B}=T_{A B} \lambda_{A B}=T_{A B} \frac{-H}{A B}=(-0.6 \mathbf{i}-0.8 \mathbf{j}) T_{A B} \\
\mathbf{T}_{A C}=T_{A C} \lambda_{A C}=T_{A C} \frac{A C}{A C}=(0.32432-0.75676 \mathbf{j}+0.56757 \mathbf{k}) T_{A C} \\
\mathbf{T}_{A D}=T_{A D} \lambda_{A D}=T_{A D} \frac{A D}{A D}=(-0.86154 \mathbf{j}-0.50769 \mathbf{k}) T_{A D}
\end{array}
$$

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## SOLUTION Continued

## Equilibrium condition

$$
\Sigma F=0: \mathbf{T}_{A B}+\mathbf{T}_{A C}+\mathbf{T}_{A D}+P \mathbf{j}=0
$$

Substituting the expressions obtained for $\mathbf{T}_{A B}, \mathbf{T}_{A C}$, and $\mathbf{T}_{A D}$ and factoring $\mathbf{i}, \mathbf{j}$, and $\mathbf{k}$ :

$$
\begin{gathered}
\left(-0.6 T_{A B}+0.32432 T_{A C}\right) \mathbf{i}+\left(-0.8 T_{A B}-0.75676 T_{A C}-0.86154 T_{A D}+P\right) \mathbf{j} \\
+\left(0.56757 T_{A C}-0.50769 T_{A D}\right) \mathbf{k}=0
\end{gathered}
$$

Equating to zero the coefficients of $\mathbf{i}, \mathbf{j}, \mathbf{k}$ :

$$
\begin{align*}
-0.6 T_{A B}+0.32432 T_{A C} & =0  \tag{1}\\
-0.8 T_{A B}-0.75676 T_{A C}-0.86154 T_{A D}+P & =0  \tag{2}\\
0.56757 T_{A C}-0.50769 T_{A D} & =0 \tag{3}
\end{align*}
$$

Setting $T_{A B}=259 \mathrm{~N}$ in (1) and (2), and solving the resulting set of equations gives

$$
\begin{aligned}
& T_{A C}=479.15 \mathrm{~N} \\
& T_{A D}=535.66 \mathrm{~N}
\end{aligned}
$$

$$
\mathbf{P}=1031 \mathrm{~N}^{\dagger}
$$



## SOLUTION

See Problem 2.112 for the figure and the analysis leading to the linear algebraic Equations (1), (2), and (3) below:

$$
\begin{array}{r}
-0.6 T_{A B}+0.32432 T_{A C}=0 \\
-0.8 T_{A B}-0.75676 T_{A C}-0.86154 T_{A D}+P=0 \\
0.56757 T_{A C}-0.50769 T_{A D}=0 \tag{3}
\end{array}
$$

Substituting $T_{A C}=444 \mathrm{~N}$ in Equations (1), (2), and (3) above, and solving the resulting set of equations using conventional algorithms gives

$$
\begin{aligned}
& T_{A B}=240 \mathrm{~N} \\
& T_{A D}=496.36 \mathrm{~N} \quad \mathbf{P}=956 \mathrm{~N}
\end{aligned}
$$



## SOLUTION

## PROBLEM 2.114

A transmission tower is held by three guy wires attached to a pin at $A$ and anchored by bolts at $B, C$, and $D$. If the tension in wire $A B$ is 630 lb , determine the vertical force $\mathbf{P}$ exerted by the tower on the pin at $A$.

## Free Body $\boldsymbol{A}$ :

$$
\begin{array}{ll}
\Sigma \mathbf{F}=0: \mathbf{T}_{A B}+\mathbf{T}_{A C}+\mathbf{T}_{A D}+P \mathbf{j}=0 \\
--\mathbb{A} B=-45 \mathbf{i}-90 \mathbf{j}+30 \mathbf{k} & A B=105 \mathrm{ft} \\
-H C=30 \mathbf{i}-90 \mathbf{j}+65 \mathbf{k} & A C=115 \mathrm{ft} \\
-\mathbb{I} & A D=110 \mathrm{ft} \\
A D=20 \mathbf{i}-90 \mathbf{j}-60 \mathbf{k} &
\end{array}
$$

We write

$$
\begin{aligned}
\mathbf{T}_{A B} & =T_{A B} \lambda_{A B}=T_{A B} \frac{-\overline{A B}}{A B} \\
& =\left(-\frac{3}{7} \mathbf{i}-\frac{6}{7} \mathbf{j}+\frac{2}{7} \mathbf{k}\right) T_{A B} \\
\mathbf{T}_{A C} & =T_{A C} \lambda_{A C}=T_{A C} \frac{--\overline{A C}}{A C} \\
& =\left(\frac{6}{23}-\frac{18}{23} \mathbf{j}+\frac{13}{23} \mathbf{k}\right) T_{A C} \\
\mathbf{T}_{A D} & =T_{A D} \lambda_{A D}=T_{A D} \frac{--I D}{A D} \\
& \left.=\left(\begin{array}{c}
2 \\
11 \\
11
\end{array}{ }^{9} \mathbf{j}-6 \frac{-}{\mathbf{k}}\right) T{ }_{11}\right)
\end{aligned}
$$

Substituting into the Eq. $\Sigma \mathbf{F}=0$ and factoring $\mathbf{i}, \mathbf{j}, \mathbf{k}$ :

$$
\begin{aligned}
& \left({ }^{1}{ }^{7} T{ }_{A B}+\frac{{ }^{\mathbf{0}}{ }_{23}}{{ }^{A C}}+{ }_{11}{ }^{2} T_{A D}\right) \mathbf{i} \\
& +\left({ }_{7}{ }^{6} T_{A B}-{ }_{23}^{18} T_{A C}-{ }_{11}^{-9} T_{A D}+P\right) \mathbf{j}
\end{aligned}
$$

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## PROBLEM 2.114 (Continued)

Setting the coefficients of $\mathbf{i}, \mathbf{j}, \mathbf{k}$, equal to zero:

$$
\begin{align*}
& \text { i: } \quad-\frac{3}{7} T_{A B}+\frac{6}{23} T{ }_{A C}+\frac{2}{11} T_{A D}=0  \tag{1}\\
& \mathbf{j}: \quad-\frac{6}{7} T_{A B}-\frac{18}{23} T_{A C}-\frac{9}{11} T_{A D}+P=0  \tag{2}\\
& \text { k: } \quad \frac{2}{7} T_{A B}+\frac{13}{23} T_{A C}-\frac{6}{11} T_{A D}=0 \tag{3}
\end{align*}
$$

Set $T_{A B}=630 \mathrm{lb}$ in Eqs. (1) - (3):

$$
\begin{align*}
-270 \mathrm{lb}+\frac{6}{23}{ }_{A C}+\frac{2}{11} T_{A D} & =0  \tag{1'}\\
-540 \mathrm{lb}-\frac{18}{23} T_{A C}-\frac{9}{11} T_{A D}+P & =0 \\
180 \mathrm{lb}+\frac{13}{23} T_{A C}-\frac{6}{11} T_{A D} & =0
\end{align*}
$$

Solving,

$$
T_{A C}=467.42 \mathrm{lb} \quad T_{A D}=814.35 \mathrm{lb} \quad P=1572.10 \mathrm{lb}
$$

$$
P=1572 \mathrm{lb}
$$



## PROBLEM 2.115

A rectangular plate is supported by three cables as shown. Knowing that the tension in cable $A D$ is 520 N , determine the weight of the plate.

## SOLUTION

See Problem 2.114 for the figure and the analysis leading to the linear algebraic Equations (1), (2), and (3) below:

$$
\begin{align*}
-\frac{8}{17} T_{A B}+0.6 T_{A C}+\frac{5}{13} T_{A D} & =0  \tag{1}\\
-\frac{12}{17} T_{A B}+0.64 T_{A C}-\frac{9.6}{13} T_{A D}+P & =0  \tag{2}\\
\frac{9}{17} T_{A B}+0.48 T_{A C}-\frac{7.2}{13} T_{A D} & =0 \tag{3}
\end{align*}
$$

Making $T_{A D}=520 \mathrm{~N}$ in Eqs. (1) and (3):

$$
\begin{align*}
& -\frac{8}{17} T_{A B}+0.6 T_{A C}+200 \mathrm{~N}=0  \tag{1'}\\
& \frac{9}{17} T_{A B}+0.48 T_{A C}-288 \mathrm{~N}=0
\end{align*}
$$

Multiply (1') by $9,\left(3^{\prime}\right)$ by 8 , and add:

$$
9.24 T_{A C}-504 \mathrm{~N}=0 T_{A C}=54.5455 \mathrm{~N}
$$

Substitute into ( $1^{\prime}$ ) and solve for $T_{A B}$ :

$$
T_{A B}=\frac{17}{8}(0.6 \times 54.5455+200) \quad T_{A B}=494.545 \mathrm{~N}
$$

Substitute for the tensions in Eq. (2) and solve for $P$ :

$$
\begin{aligned}
P & =\frac{12}{17}(494.545 \mathrm{~N})+0.64(54.5455 \mathrm{~N})+\frac{9.6}{13}(520 \mathrm{~N}) & \\
& =768.00 \mathrm{~N} & \text { Weight of plate }=P=768 \mathrm{~N}
\end{aligned}
$$

